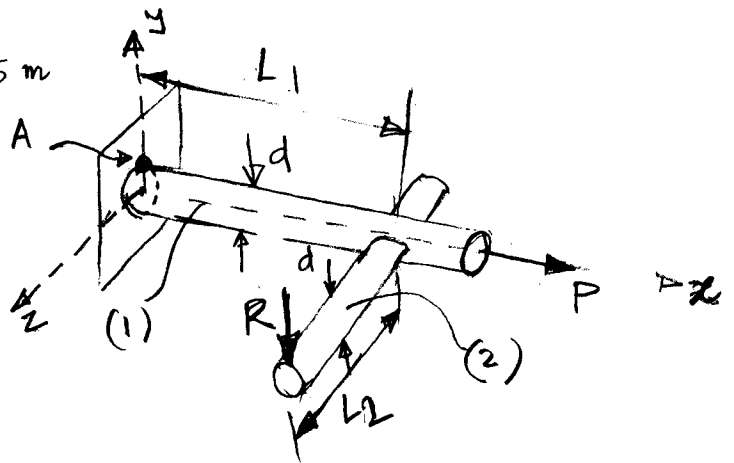


Problem - 1

Calculate the principal stresses generated at A

Given,  $L_1 = 1.5 \text{ m}$  }  $d = 0.05 \text{ m}$   
 $L_2 = 1.0 \text{ m}$   
 $R = 500 \text{ N}$   
 $P = 100 \text{ kN}$



Solution

We need to create the stress block at A

(1) Consider the effect of load P at A

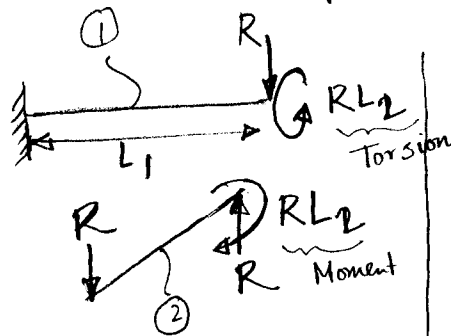
It will produce only axial stress  $\sigma_x = \frac{P}{A}$

$$\text{i.e. } \sigma_x^{(a)} = \frac{P}{A} = \frac{100(10)^3}{\frac{\pi(0.05)^2}{4}} = 50.93(10)^6 \text{ N/m}^2 = \underline{\underline{50.93 \text{ MPa}}}$$

(2) Consider the effect of load R at A now,

It has two effects (a) bending (b) twisting!

Draw the free bodies of beam (1) & (2)



This was discussed in the class before.

At the junction Newton's 3rd law follows.

Moment ( $RL_2$ ) in beam (2) is transferred as a twist of same magnitude ( $RL_2$ ) in beam (1)

Hence, moment at A,  $M = -RL_1$

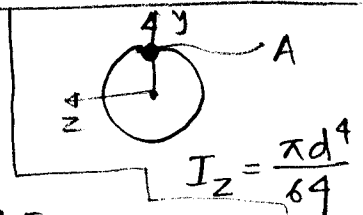
Torsion at A,  $T = RL_2$

(a) Moment will induce bending (flexural)

$$\text{stress } \sigma_x^{(b)} = -\frac{My}{I_z} = -\frac{(-RL_1) \frac{d}{2}}{\frac{\pi d^4}{64}}$$

Note: It's a tensile stress. since beam bends downwards, top fiber stretches! Reasonable

$$= \frac{32RL_1}{\pi d^3} = \frac{32(500)1.5}{\pi(0.05)^3} = 61.12(10)^6 \text{ N/m}^2 = \underline{\underline{61.12 \text{ MPa}}}$$



(b) Torsion will induce shear stress

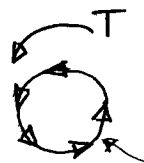
$$\tau = \frac{T r}{I_p}$$

$$= \frac{(R L_2)(d/2)}{\left(\frac{\pi d^4}{32}\right)} = \frac{16 R L_2}{\pi d^3}$$

$$= \frac{16(500)(1)}{\pi(0.05)^3}$$

$$= 20.37 (10)^6 \text{ N/m}^2$$

$$= 20.37 \text{ MPa}$$

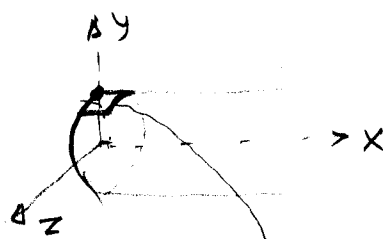


flow of shear stress on the cross section

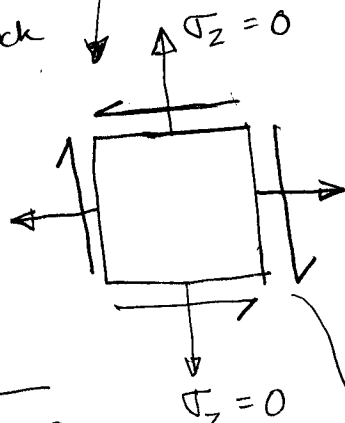
[This determines the direction of  $\tau$  in the stress block]

Note: in the stress block it is negative according to the sign convention

We need to draw the stress block at A in the x-z plane, but the problem remains 2D. We will just use 'z' as subscript everywhere instead of y.



stress block in x-z plane



Superposition

$$\sigma_x = \sigma_x(a) + \sigma_x(b)$$

$$= 50.93 + 61.12$$

$$= 112.05 \text{ MPa}$$

$$\tau_{xz} = -\tau = -20.37 \text{ MPa}$$

Principal stresses:

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_z}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_z}{2}\right)^2 + \tau_{xz}^2}$$

$$= \frac{112.05}{2} \pm \sqrt{\left(\frac{112.05}{2}\right)^2 + (-20.37)^2}$$

$$= 56.025 \pm 59.613, \quad \sigma_1 = 115.6 \text{ MPa}$$

$$\sigma_2 = -3.6 \text{ MPa}$$

\* You could also calculate  $\tau_{max}$ ,  $\epsilon_1$ ,  $\epsilon_2$ ,  $\gamma_{max}$ , show them in properly oriented elements.

\* You can also check the safety at point A, i.e., whether,  $|\sigma_1|$  &  $|\sigma_2| \leq \frac{\sigma_y}{f_s}$  yield stress of material in tension/compression &  $|\tau_{max}| \leq \frac{\tau_y}{f_s}$  yield stress in shear