

DEPARTMENT OF CIVIL ENGINEERING, IIT BOMBAY
CE-317 STRUCTURAL MECHANICS II
 Endsem exam 18/11/11

Notes

1. Common data for problems 2, 3, 4: $E = 200 \text{ GPa}$, $G = 80 \text{ GPa}$, $I_y = I_z = 8000 \text{ cm}^4$, $A = 200 \text{ cm}^2$, $J = 10000 \text{ cm}^4$
2. Numerical answers must be accurate to 2 or more places after decimal.
3. Must: use member end (local) coordinate system as done in class; use convention for all forces and displacements (linear and angular) as in class; use numbering sequence of structure's nodal forces and displacements as in class.

Problem 1

Data: $EI_z = 10$, $A/I_z = 5$, $L = 2$, $P = 1$. Rotational spring constant $k = 5$. Settlement of support node 3 is 0.1m downward. Member 23 is heated 20°C above ambient temperature, coefficient of linear expansion is $0.000012/^\circ\text{C}$. All data provided in units of N, m.

Find the numerical values of the stiffness matrix \mathbf{K}_{II} and the load vector \mathbf{P}_I that are required to solve for the displacements. You do not need to invert \mathbf{K}_{II} and do not need to solve for displacements using $\mathbf{K}_{II}^{-1}\mathbf{P}_I$. Settlement must be handled by including it in load vector \mathbf{P}_I .

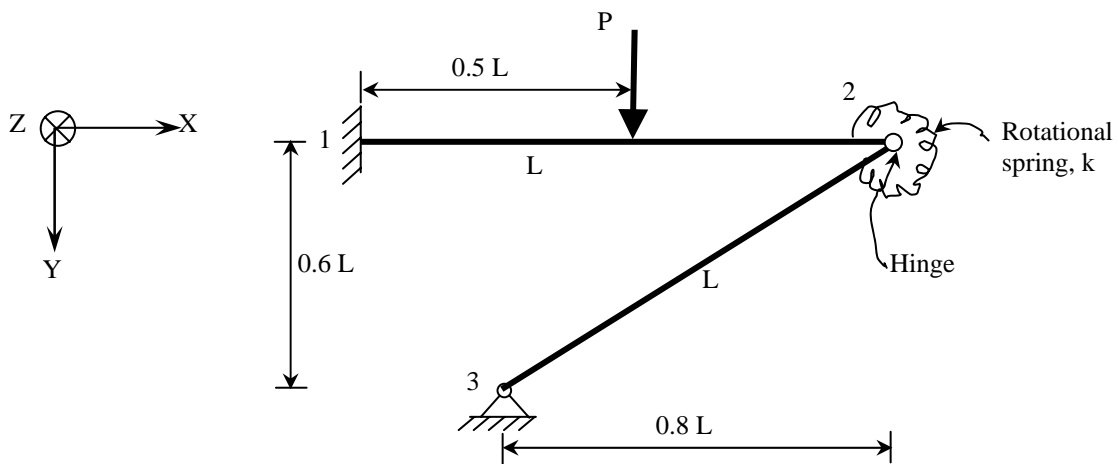


Fig. 1

Problem 2

Taking the redundant at node 4. Use Flexibility matrix method to find redundant.

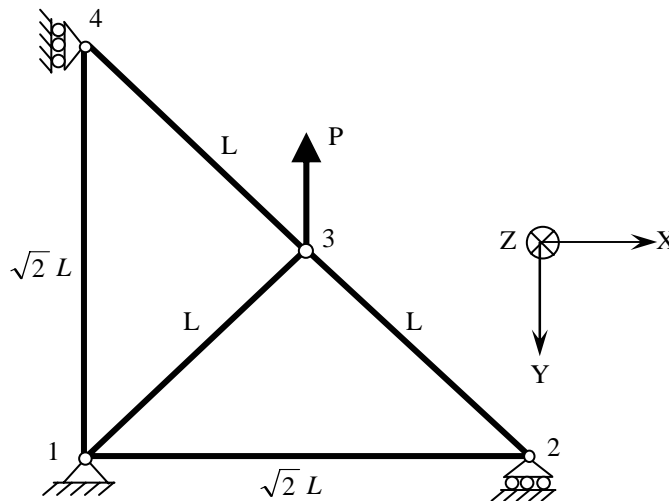


Fig. 2

Problem 3

Data: $L = 10\text{ m}$

Plan view of a grid is shown in **Fig. 3**. Thus, the vertical downward direction (i.e., Z) acts into the plane of the paper. All 16 members of the grid are subjected to vertically downward uniformly distributed load of magnitude **20 kN/m**. Further, the grid supports a vertically downward load of magnitude **100 kN** at joint 1.

Find deflections at joint 1 and reactions at joint 2 using Stiffness Method.

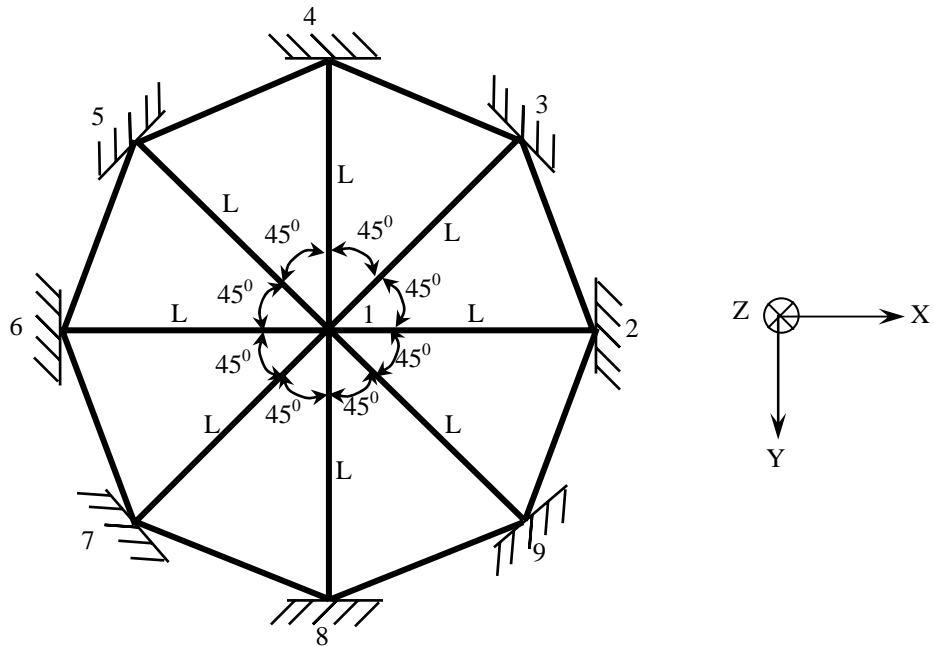


Fig. 3

Problem 4

Data: $L = 10\text{ m}$

Find deflections at joint 2 using Stiffness Method.

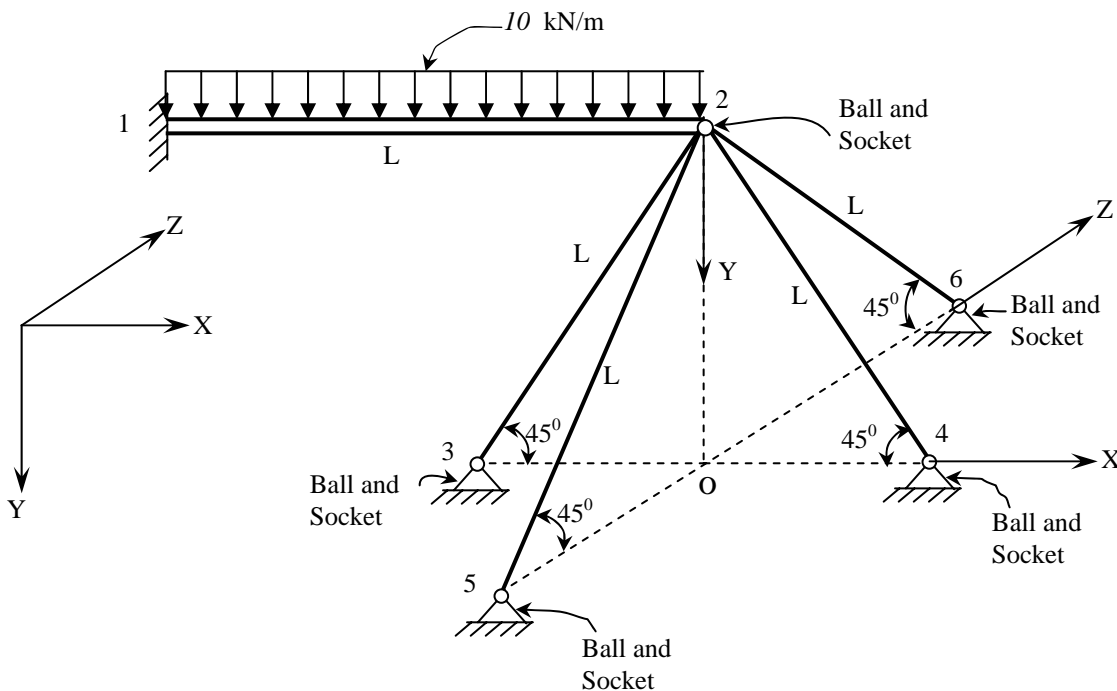
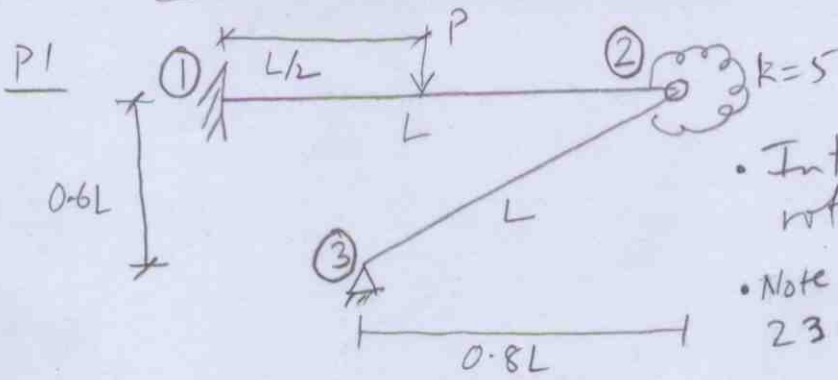


Fig. 4



- Internal hinge with partial rotational restraint.
- Note that due to spring, member 23 behaves as frame & not truss.

$$a_{21} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad a_{23} = \begin{bmatrix} 0.8 & -0.6 & 0 \\ 0.6 & 0.8 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

• Total 10 displs (3 at ①, 4 at ②, 3 at ③)
• dof's are (4, 5, 6, 7, 10)

$$K_{22} (4 \times 4), \quad K_{23} (4 \times 3), \quad K_{32} (3 \times 4), \quad K_{33} (3 \times 3)$$

$$K_{II} = \left[\begin{array}{c|c} K_{22} & K_{23} \text{ (last col)} \\ \hline K_{32} \text{ (last row)} & K_{33} (3,3) \end{array} \right] \rightarrow \text{not required for element wise assembly}$$

$$k_{22}^1 = k_{22}^2 = k_{33}^3 = \frac{EI}{L} \begin{bmatrix} A/I & 0 & 0 \\ 0 & 12/L^2 & -6/L \\ 0 & -6/L & 4 \end{bmatrix} = 5 \begin{bmatrix} 5 & 0 & 0 \\ 0 & 3 & -3 \\ 0 & -3 & 4 \end{bmatrix}; \quad k_{23}^2 = 5 \begin{bmatrix} 5 & 0 & 0 \\ 0 & 3 & -3 \\ 0 & -3 & 2 \end{bmatrix}$$

$$K_{22}^1 = a_{21}^T k_{22}^1 a_{21} = k_{22}^1 = 5 \begin{bmatrix} 5 & 0 & 0 \\ 0 & 3 & -3 \\ 0 & -3 & 4 \end{bmatrix} \begin{matrix} 4 \\ 5 \\ 6 \end{matrix}$$

$$K_{22}^3 = a_{23}^T k_{22}^3 a_{23} = 5 \begin{bmatrix} 4 & 1.8 & -1.8 \\ -3 & 2.4 & -2.4 \\ 0 & -3 & 4 \end{bmatrix} \begin{bmatrix} 0.8 & -0.6 & 0 \\ 0.6 & 0.8 & 0 \\ 0 & 0 & 1 \end{bmatrix} = 5 \begin{bmatrix} 4.28 & -0.96 & -1.8 \\ -0.96 & 3.72 & -2.4 \\ -1.8 & -2.4 & 4 \end{bmatrix} \begin{matrix} 4 \\ 5 \\ 7 \end{matrix}$$

$$K_{22}^{\text{spring}} = 5 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{matrix} 6 \\ 7 \end{matrix}$$

$$a_{23}^T K_{23} a_{32} = \hat{K}_{23} = 5 \begin{bmatrix} 4 & 1.8 & -1.8 \\ -3 & 2.4 & -2.4 \\ 0 & -3 & 2 \end{bmatrix} \begin{bmatrix} -0.8 & 0.6 & 0 \\ -0.6 & -0.8 & 0 \\ 0 & 0 & 1 \end{bmatrix} = 5 \begin{bmatrix} -4.28 & 0.96 & -1.8 \\ 0.96 & -3.72 & -2.4 \\ 1.8 & 2.4 & 2 \end{bmatrix} \begin{matrix} 8 \\ 9 \\ 10 \end{matrix}$$

$$\hat{K}_{32} = a_{32}^T K_{32} a_{23} = \begin{bmatrix} K_{23}^T \\ 0 \\ 0 \end{bmatrix} \begin{matrix} 8 \\ 9 \\ 10 \end{matrix}$$

$$K_{33} = a_{32}^T k_{33}^2 a_{32} = 5 \begin{bmatrix} -4 & -1.8 & 1.8 \\ 3 & -2.4 & 2.4 \\ 0 & -3 & 4 \end{bmatrix} \begin{bmatrix} -0.8 & 0.6 & 0 \\ -0.6 & -0.8 & 0 \\ 0 & 0 & 1 \end{bmatrix} = 5 \begin{bmatrix} 4.28 & -0.96 & 1.8 \\ -0.96 & 3.72 & 2.4 \\ 1.8 & 2.4 & 4 \end{bmatrix} \begin{matrix} 8 \\ 9 \\ 10 \end{matrix}$$

Node-wise assembly

From above \hat{K}_{23} ,

$$K_{23} = 5 \begin{bmatrix} -4.28 & 0.96 & -1.8 \\ 0.96 & -3.72 & -2.4 \\ 0 & 0 & 0 \\ 1.8 & 2.4 & 2 \end{bmatrix} \begin{matrix} 4 \\ 5 \\ 6 \\ 7 \end{matrix}$$

$$K_{32} = K_{23}^T$$

$$K_{22} = \left[\begin{array}{c|ccc} K_{22}^1 & 0 & & \\ \hline & 0 & & \\ & 0 & & \\ & 0 & 0 & 0 \end{array} \right] + \left[\begin{array}{c|ccc} K_{22}^3 & & & \\ \hline & & & \\ & & & \\ & & & \end{array} \right] + 5 \left[\begin{array}{cccc} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & -1 \\ 0 & 0 & -1 & 1 \end{array} \right]$$

contribution of mem 1-2 insert 2nd last, ie 3rd, row {0000} contribution of spring
 contribution of mem 2-3

$$K_{22} = 5 \begin{bmatrix} 5+4.28+0 & 0-0.96+0 & 0+0+0 & 0-1.8+0 \\ 0-0.96+0 & 3+3.72+0 & -3+0+0 & 0-2.4+0 \\ 0+0+0 & -3+0+0 & 4+0+1 & 0+0-1 \\ 0-1.8+0 & 0-2.4+0 & 0+0-1 & 0+4+1 \end{bmatrix}$$

$$K_{II} = \begin{bmatrix} 46.4 & -4.8 & 0 & -9 & -9 \\ -4.8 & 33.6 & -15 & -12 & -12 \\ 0 & -15 & 25 & -5 & 0 \\ -9 & -12 & -5 & 25 & 10 \\ -9 & -12 & 0 & 10 & 20 \end{bmatrix}$$

Element-wise assembly - Direct stiffness method

Add element contributions consistently using row-col node number pairs. Only those element submatrices will participate, at least one entry whose row & col numbers correspond to dof's (ie unconstrained nodes). Thus

$$K_{II} = K_{22}^1 + K_{22}^3 + K_{22}^{spring} + \hat{K}_{23} + \hat{K}_{32} + K_{33}, \text{ added consistently.}$$

	4	5	6	7	10	
5	5+4.28	-0.96	-	-1.8	-1.8	4
	-0.96	3+3.72	-3	-2.4	-2.4	5
	-	-3	4+1	-1	-	6
	-1.8	-2.4	-1	4+1	2	7
	-1.8	-2.4	-	2	4	8

((-) dashes are zeros) → same as above result

Loads

$$P^a = \underline{0}; \quad \tilde{P}_M^e = \{0 \quad -P/2 \quad PL/8 \quad 0 \quad 0\}^T$$

(3)

cut at (3) for temp, \therefore settlement given at (3).

$$\{\delta_s\}_3 = \begin{Bmatrix} -\alpha T \nu L \\ 0 \\ 0 \end{Bmatrix} + \begin{matrix} \begin{bmatrix} -0.8 & 0.6 & 0 \\ -0.6 & -0.8 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{Bmatrix} 0 \\ 0.1 \\ 0 \end{Bmatrix} \\ \begin{matrix} \uparrow \\ a_{32} \end{matrix} \quad \begin{matrix} \uparrow \\ \text{settlement} \end{matrix} \end{matrix} = \begin{Bmatrix} -4.8E-4 + 0.06 \\ -0.08 \\ 0 \end{Bmatrix}$$

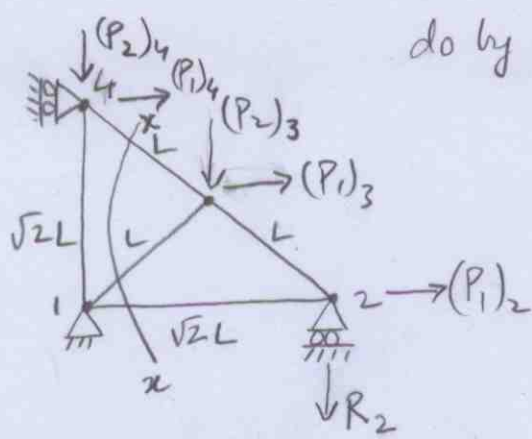
$$\{F_s\}_{32}^f = K_{33}^2 \{\delta_s\}_3 = 5 \begin{bmatrix} 5 & 0 & 0 \\ 0 & 3 & -3 \\ 0 & -3 & 4 \end{bmatrix} \begin{Bmatrix} 0.05952 \\ -0.08 \\ 0 \end{Bmatrix} = 5 \begin{Bmatrix} 0.2976 \\ -0.24 \\ 0.24 \end{Bmatrix}$$

$$\{F_s\}_{23}^f = K_{23} \{\delta_s\}_3 = 5 \begin{bmatrix} 5 & 0 & 0 \\ 0 & 3 & -3 \\ 0 & -3 & 2 \end{bmatrix} \begin{Bmatrix} 0.05952 \\ -0.08 \\ 0 \end{Bmatrix} = 5 \begin{Bmatrix} 0.2976 \\ -0.24 \\ 0.24 \end{Bmatrix}$$

$$\begin{aligned} \{P_s\}_3^e &= a_{32}^T \{F_s\}_{32}^f = 5 \begin{Bmatrix} - \\ - \\ 0.24 \end{Bmatrix} \\ \{P_s\}_2^e &= a_{23}^T \{F_s\}_{23}^f = 5 \begin{Bmatrix} 0.09408 \\ -0.37056 \\ 0.24 \end{Bmatrix} \end{aligned} \Rightarrow \tilde{P}_S^e = 5 \begin{Bmatrix} 0.09408 \\ -0.37056 \\ 0 \\ 0.24 \\ 0.24 \end{Bmatrix}$$

$$P_I = P^a - \tilde{P}_M^e - \tilde{P}_S^e = \begin{Bmatrix} -0.4704 \\ 2.3528 \\ -0.25 \\ -1.2 \\ -1.2 \end{Bmatrix}$$

P2



do by Flexibility matrix method.

(4)

$$P_{II} = \{(P_1)_4\}$$

$$P_I = \{(P_1)_2, (P_1)_3, (P_2)_2, (P_2)_4\}$$

Equilibrium matrices

$$\sum M_{\odot} = 0 \Rightarrow R_2 \sqrt{2}L + (P_1)_3 \frac{L}{\sqrt{2}} + (P_2)_3 \frac{L}{\sqrt{2}} + (P_1)_4 \sqrt{2}L = 0$$

$$F_{32} = F_{23} = R_2 \sqrt{2} = - (P_1)_3 \frac{1}{\sqrt{2}} - (P_2)_3 \frac{1}{\sqrt{2}} - (P_1)_4 \sqrt{2}$$

$$F_{21} = F_{12} = (P_1)_2 - \frac{F_{23}}{\sqrt{2}} = (P_1)_2 + (P_1)_3 \cdot \frac{1}{2} + (P_2)_3 \frac{1}{2} + (P_1)_4$$

$$\text{Section } x-x \rightarrow \sum M_{\odot} = 0 \Rightarrow F_{31} \cdot L - (P_1)_3 \frac{L}{\sqrt{2}} + (P_2)_3 \frac{L}{\sqrt{2}} = 0$$

$$= F_{13} \Rightarrow F_{31} = F_{13} = (P_1)_3 \frac{1}{\sqrt{2}} - \frac{(P_2)_3}{\sqrt{2}}$$

$$\sum M_{\odot} = 0 \Rightarrow F_{43} \cdot L + (P_1)_4 \sqrt{2}L = 0 \Rightarrow F_{34} = F_{43} = -\sqrt{2}(P_1)_4$$

$$= F_{34}$$

$$F_{41} = F_{14} = - (P_2)_4 - F_{43} \cdot \frac{1}{\sqrt{2}} = - (P_2)_4 + (P_1)_4$$

$$\begin{Bmatrix} F_{21} \\ F_{32} \\ F_{31} \\ F_{43} \\ F_{41} \end{Bmatrix} = \underbrace{\begin{bmatrix} 1 & 1/2 & 1/2 & 0 & 0 \\ 0 & -1/\sqrt{2} & -1/\sqrt{2} & 0 & 0 \\ 0 & 1/\sqrt{2} & -1/\sqrt{2} & 0 & 0 \\ 0 & 0 & 0 & 0 & -1 \\ 0 & 0 & 0 & 0 & 1 \end{bmatrix}}_{b_I} \underbrace{\begin{bmatrix} 1 \\ -\sqrt{2} \\ 0 \\ -\sqrt{2} \\ 1 \end{bmatrix}}_{b_{II}} \begin{Bmatrix} (P_1)_2 \\ (P_1)_3 \\ (P_2)_3 \\ (P_2)_4 \\ (P_1)_4 \end{Bmatrix}$$

$\left. \begin{matrix} (P_1)_2 \\ (P_1)_3 \\ (P_2)_3 \\ (P_2)_4 \end{matrix} \right\} \rightarrow P_I$
 $\left. \begin{matrix} (P_1)_4 \end{matrix} \right\} \rightarrow P_{II}$

$$du = \frac{L}{EA} \begin{bmatrix} \sqrt{2} & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & \sqrt{2} \end{bmatrix}$$

(5)

$$P_{II} = -D_{II II}^{-1} D_{II I} P_I$$

$$D_{II II} = b_{II}^T d_u b_{II} = \frac{L}{EA} \{ \sqrt{2} \quad -\sqrt{2} \quad 0 \quad -\sqrt{2} \quad \sqrt{2} \} b_{II}$$

$$= \sqrt{2} + 2 + 2 + \sqrt{2} = 4 + 2\sqrt{2}$$

$$D_{II I} = b_{II}^T d_u b_I = \{ \sqrt{2} \quad 1 + \frac{1}{\sqrt{2}} \quad 1 + \frac{1}{\sqrt{2}} \quad -\sqrt{2} \} \frac{L}{EA}$$

$$P_{II} = - \frac{1}{(4 + 2\sqrt{2})} \left\{ \sqrt{2} \quad 1 + \frac{1}{\sqrt{2}} \quad 1 + \frac{1}{\sqrt{2}} \quad -\sqrt{2} \right\} \begin{Bmatrix} 0 \\ 0 \\ -P \\ 0 \end{Bmatrix} = \frac{\sqrt{2} + 1}{(4\sqrt{2} + 4)} P = \frac{P}{4}$$

P3 Grid $\Rightarrow \{(\Delta_1)_1, (\Delta_2)_1, (\Delta_6)_1\} = \{0 \ 0 \ 0\}$

Symmetry $\Rightarrow \{(\Delta_4)_1, (\Delta_5)_1\} = \{0 \ 0\}$

ie no out-of-plane rotation of joint 1, due to symmetry.

So only $(\Delta_3)_1$ is $\neq 0$, ie ^{only} one-dof activated.

Always, $K_{II} = \sum_{i=1}^9 K_{II}^i$

ie K_{II} contains entries pertaining to dofs (3,4,5) at node 1. But \because dofs (4,5) are known zero from symmetry, we need only $K_{II}(1,1)$ to solve for $(\Delta_3)_1$.

Symmetry, ie 1-dof, $\Rightarrow K_{II} = K_{II}(1,1)$

eg $K_{II}^3(1,1) = a_{13}^T K_{II}^3 a_{13}$, $a_{13} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -\frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} \\ 0 & -\frac{1}{\sqrt{2}} & -\frac{1}{\sqrt{2}} \end{bmatrix}$, $K_{II}^3 = \frac{EI}{L} \begin{bmatrix} 12/L^2 & 0 & 6/L \\ 0 & 0.5 & 0 \\ 6/L & 0 & 4 \end{bmatrix}$

where we used $\frac{GJ}{EIy} = \frac{80 \times 10000}{200 \times 8000} = 0.5$

$\Rightarrow K_{II}^3(1,1) = \frac{EI}{L} \frac{12}{L^2} \Rightarrow K_{II} = K_{II}(1,1) = 8 \times K_{II}^3(1,1)$
 $= 8 \times \frac{12}{L^2} \frac{EIy}{L} = \frac{96}{10^2} \frac{200 \times 10^6 \times 8000}{10} = 1536$

$K_{II} = 1536$

$P_I = [100 + \frac{20 \times 10}{2} \times 8] = [900] \text{ kN}$

$\Delta_I = \{(\Delta_3)_1\} = \frac{1}{1536} \cdot 900 = 0.5859 \text{ m}$

$F_{21} = K_{22}^1 \delta_2 + K_{21} \delta_1 + \left\{ \begin{array}{l} -wL \\ 0 \\ -\frac{wL^2}{12} \cos(67.5) \times 2 \end{array} \right\} + \left\{ \begin{array}{l} -wL/2 \\ 0 \\ -wL^2/12 \end{array} \right\}$

Fef's of circumferential members, ie $\{F_{23}^F + F_{29}^F\}$

$= \frac{EIy}{L} \begin{bmatrix} -12/L^2 & 0 & -6/L \\ 0 & 0.5 & 0 \\ -6/L & 0 & -2 \end{bmatrix} \begin{Bmatrix} 900/1536 \\ 0 \\ 0 \end{Bmatrix} + \left\{ \begin{array}{l} -20 \times 2 \times 10 \cos(67.5) \\ 0 \\ -20 \times \frac{(2 \times 10 \cos(67.5))^2}{12} \end{array} \right\}$

$= 1600 \begin{bmatrix} -0.12 & 0 & -0.6 \\ 0 & 0.5 & 0 \\ -0.6 & 0 & -2 \end{bmatrix} \begin{Bmatrix} 900/1536 \\ 0 \\ 0 \end{Bmatrix} + \begin{Bmatrix} -153.07 \\ 0 \\ -74.72 \end{Bmatrix} + \begin{Bmatrix} -100 \\ 0 \\ -166.67 \end{Bmatrix} + \begin{Bmatrix} -20 \times 10/2 \\ 0 \\ -20 \times 10^2/12 \end{Bmatrix}$

NOTE: Symmetry of structure & loading $\Rightarrow (\Delta_4)_1 = (\Delta_5)_1 = 0$, ie when $(P_4)_1 = (P_5)_1 = 0$ & $(P_3)_1 \neq 0$
 Thus K_{II} is diagonal for this symmetric structure.
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P4 Symmetry $\Rightarrow (\Delta_3)_2 = (\Delta_4)_2 = (\Delta_5)_2 = 0$ at joint 2, ie (7)
 members 12, 23, 24 deform in XY plane only. Actually
 as we see later even 25, 26 deform in XY plane.

Thus,

$$a_{21} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}; \quad a_{23} = \left\{ \frac{1}{\sqrt{2}} \quad -\frac{1}{\sqrt{2}} \quad 0 \right\}; \quad a_{24} = \left\{ -\frac{1}{\sqrt{2}} \quad -\frac{1}{\sqrt{2}} \quad 0 \right\}$$

$$a_{25} = \left\{ 0 \quad -\frac{1}{\sqrt{2}} \quad \frac{1}{\sqrt{2}} \right\}; \quad a_{26} = \left\{ 0 \quad -\frac{1}{\sqrt{2}} \quad -\frac{1}{\sqrt{2}} \right\}$$

$$K_{22}^1 = \frac{EI_z}{L} \begin{bmatrix} A/I_z & 0 & 0 \\ 0 & 12/L^2 & -6/L \\ 0 & -6/L & 4 \end{bmatrix} = 1600 \begin{bmatrix} 250 & 0 & 0 \\ 0 & 0.12 & -0.6 \\ 0 & -0.6 & 4 \end{bmatrix}, \quad \text{where we used}$$

$$\frac{EI_z}{L} = \frac{200 E 6 \cdot 8000 E^{-8}}{10} = 1600$$

$$K_{22}^3 = K_{22}^4 = K_{22}^5 = K_{22}^6 = [EA/L] = [4ES]$$

$$\frac{A}{I_z} = \frac{200 E^{-4}}{8000 E^{-8}} = 250$$

$$K_{22}^1 = a_{21}^T K_{22}^1 a_{21} = K_{22}^1 = 1600 \begin{bmatrix} 250 & 0 & 0 \\ 0 & 0.12 & -0.6 \\ 0 & -0.6 & 4 \end{bmatrix}$$

$$K_{22}^3 = a_{23}^T K_{22}^3 a_{23} = \begin{bmatrix} 1/\sqrt{2} \\ -1/\sqrt{2} \\ 0 \end{bmatrix} [4ES] \begin{bmatrix} 1/\sqrt{2} & -1/\sqrt{2} & 0 \end{bmatrix} =$$

$$\begin{bmatrix} 2ES & -2ES & 0 \\ -2ES & 2ES & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \end{matrix}$$

$$K_{22}^4 = \begin{bmatrix} 1 & 2 & 3 \\ 2ES & 2ES & 0 \\ 2ES & 2ES & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \end{matrix}$$

$$K_{22}^5 = \begin{bmatrix} 1 & 2 & 3 \\ 0 & 0 & 0 \\ 0 & 2ES & -2ES \\ 0 & -2ES & 2ES \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \end{matrix}$$

$$K_{22}^6 = \begin{bmatrix} 1 & 2 & 3 \\ 0 & 0 & 0 \\ 0 & 2ES & 2ES \\ 0 & 2ES & 2ES \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \end{matrix}$$

$$K_{22} = \begin{bmatrix} 1 & 2 & 6 \\ 4ES+2ES+2ES & -2ES+2ES & 0 \\ -2ES+2ES & 192+2ES+2ES & -960 \\ 0 & -960 & 6400 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 6 \end{matrix} = K_{FI} = \begin{bmatrix} 8ES & 0 & 0 \\ 0 & (8ES+192) & -960 \\ 0 & -960 & 6400 \end{bmatrix}$$

$$P_I = \begin{Bmatrix} 0 \\ 10 \times 10 / 2 \\ -10 \times 10^2 / 12 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 50 \\ -1000 / 12 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 50 \\ -83.33 \end{Bmatrix}$$

$$\Delta_I = K_{II}^{-1} P_I = \begin{bmatrix} 1/8ES & 0 & 0 \\ 0 & \frac{1}{\det} \begin{bmatrix} 6400 & 960 \\ 960 & (8ES+192) \end{bmatrix} \\ 0 & \frac{1}{\det} \begin{bmatrix} 960 & (8ES+192) \end{bmatrix} \end{bmatrix} \begin{Bmatrix} 0 \\ 50 \\ -1000 / 12 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 240000 \\ 6663466.67 \end{Bmatrix} \frac{1}{\det}$$

$$\det = 5120307200$$

$$= \begin{Bmatrix} 0 \\ 4.69E-5 \\ 0.01301 \end{Bmatrix} \Rightarrow (\Delta_2)_1 = 0 \Rightarrow \text{can use beam formulation and } 2 \times 2 \text{ transf for truss members.}$$

and no X-dir loading
 on 4 truss members, so no X-dir mem force in mem 21, ie no X-dir reactions at nodes 1, 3, 4, 5, 6, \Rightarrow no X-dir displ at node 2.
 This is expected \because beam 12 is just resting