

**Problem 1**

**Use only Matrix Stiffness Method.**

Members of a raft foundation are modeled as the rigid jointed grid lying in the horizontal plane as shown in **Fig. 1**. The grid consists of two orthogonal members having identical length, properties and loading. The grid is subjected to vertical soil loads and carries a vertical column load of 500 kN at **O**. A pile, which is modeled as a vertical spring, provides support to the grid at **O**.

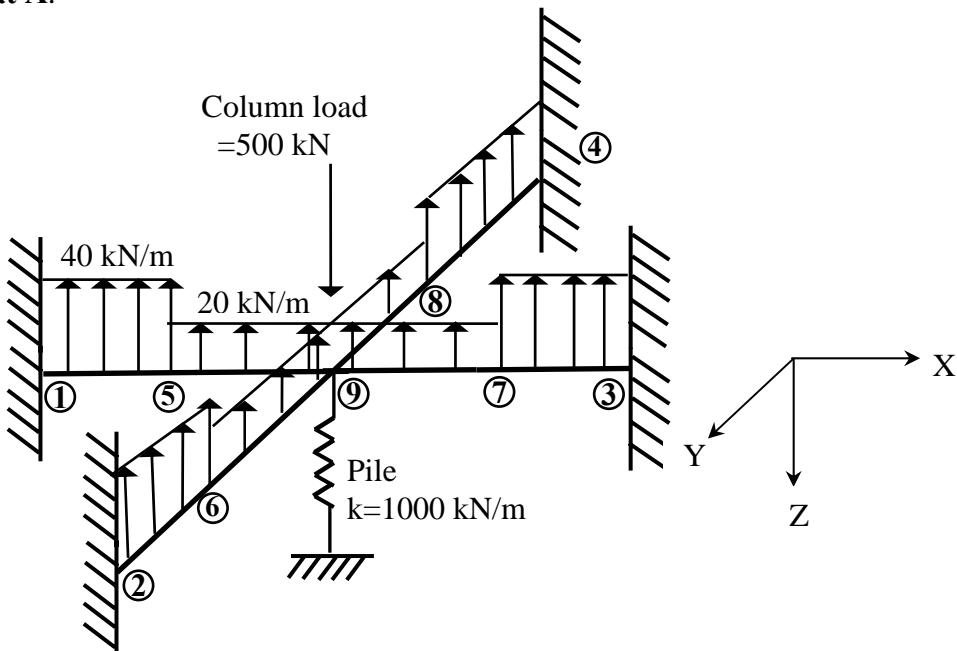
Data:  $AM = MO = OP = PC = BN = NO = OQ = QD = 2$  m.

40 kN/m acts on  $AM, PC, BN, QD$  portions of the members

20 kN/m acts on  $MO, OP, NO, OQ$  portions of the members

**Find: Reactions at A.**

**Fig. 1**



**Problem 2**

**Use only Matrix Stiffness Method.**

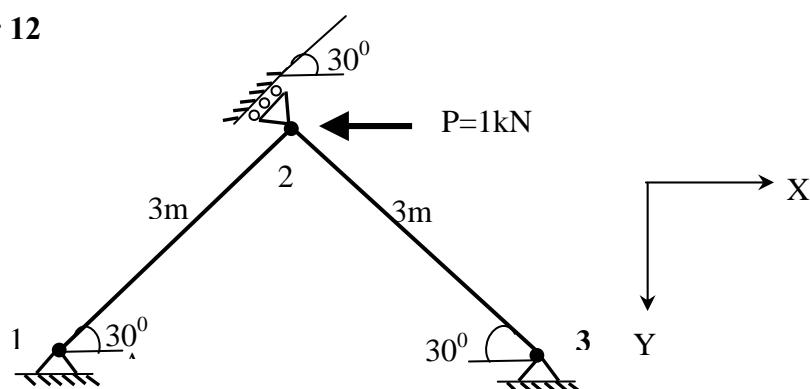
Data:  $EI_y = EI_z = 9$ ,  $A/I_y = 2/3$ ,  $L = 3$ ,  $GJ/EI_y = 0.5$ ,  $P = 2$ .

Settlement of support node 3: 0.2m downward.

Member 12 is heated 20 °C above ambient temperature, coefficient of linear expansion is 0.01/°C.

**Find: Force in member 12**

**Fig. 2**



### Problem 3

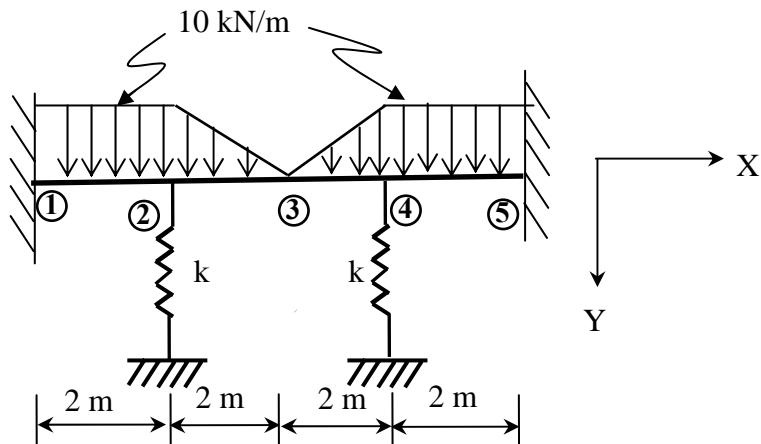
Use only Stiffness Matrix Method.

Data: vertical displacement of  $B$  measured as 1.909 mm;

$$k = 5000 \text{ kN/m}; EI = 10000 \text{ kN.m}^2$$

Find: Vertical displacement at  $C$  and rotation at  $D$

Fig. 3



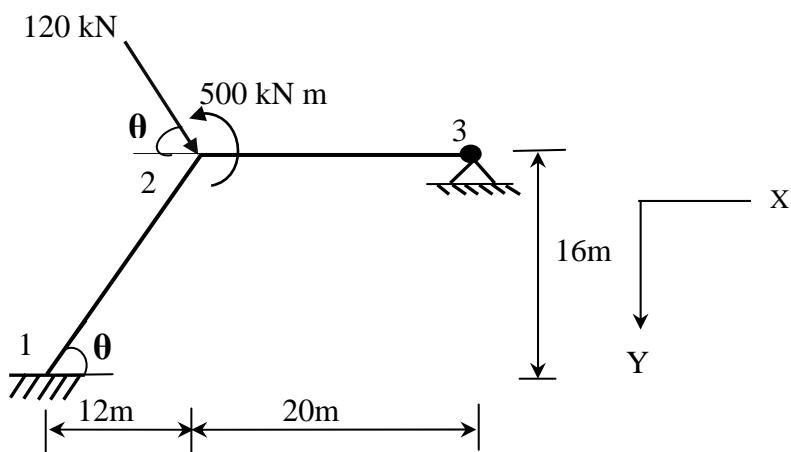
### Problem 4

Use only Flexibility Matrix Method.

Data:  $EI / EA = 100 \text{ cm}^2$

Find: Reactions at node 3

Fig. 4



P1 Symmetry  $\Rightarrow (\Delta_4)_q = (\Delta_5)_q = 0$  (ie  $(\theta_x)_q = (\theta_y)_q = 0$ )

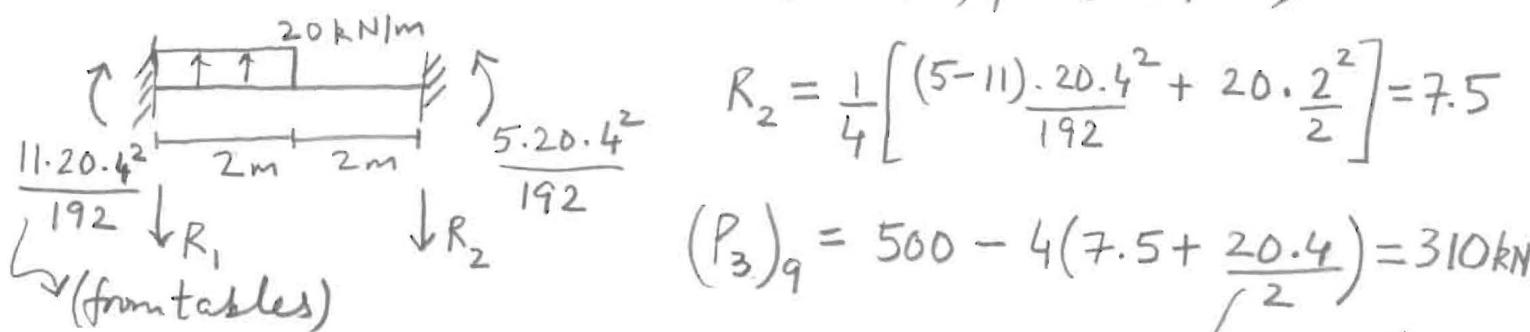
$K_{II} = K_{qq}$  ( $3 \times 3$  matrix).

$\therefore (\Delta_4)_o = (\Delta_5)_o = 0$ , we need only  $K_{qq}(1,1)$  to find  $(\Delta_3)_q$

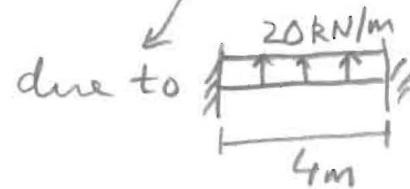
$\therefore a_{ij} = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \square & 0 \\ 0 & 0 & 1 \end{pmatrix}$  for grid,  $K_{qq}(1,1) = \sum_{j=1}^4 R_{qj}(1,1) + k$

$$K_{qq}(1,1) = K_{II}(1,1) = \frac{12EI}{L^3} * 4 + k = \frac{12 \cdot 200E6 \cdot 10^{-3}}{16} + 1000 = 151000 \text{ KN/m}$$

$$(P_3)_q = K_{II}(1,1)(\Delta_3)_q \quad (\because (\Delta_4)_q = (\Delta_5)_q = 0)$$



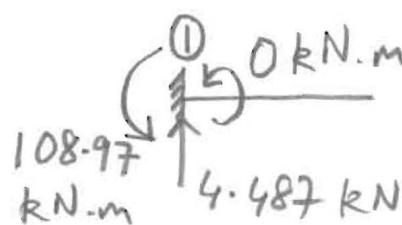
$$(\Delta_3)_q = \frac{310}{151000} \text{ m} = 2.053 \text{ mm}$$



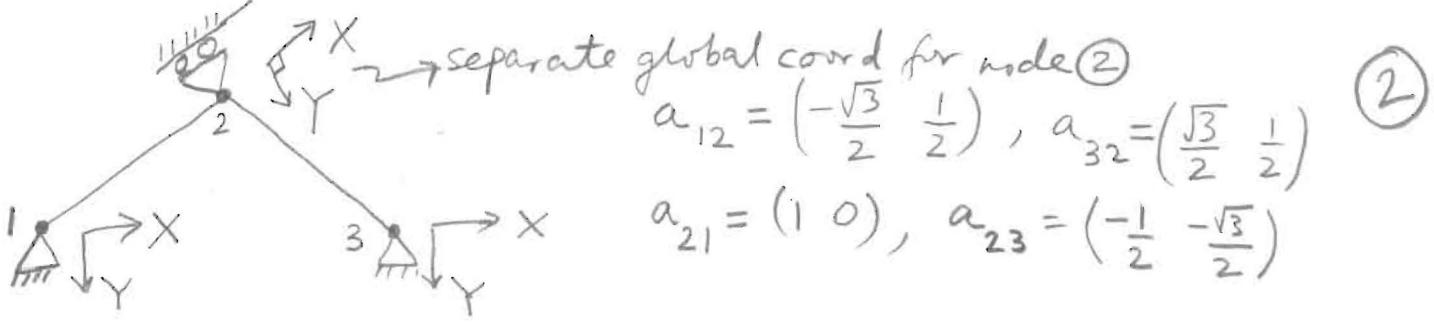
$$F_{1q} = R_{1q} \delta_{q1} + F_{1q}^f \quad (\because \delta_{1q} = 0)$$

$$= \frac{EI}{L} \begin{bmatrix} -12/L^2 & 0 & -6/L \\ 0 & GJ/EI & 0 \\ -6/L & 0 & -2 \end{bmatrix} \begin{Bmatrix} 2.053 \cdot 10^{-3} \\ 0 \\ 0 \end{Bmatrix} + \begin{Bmatrix} \frac{20 \cdot 4}{2} + (20 \cdot 2 - 7.5) \\ 0 \\ \frac{11 \cdot 20 \cdot 4^2}{192} + \frac{20 \cdot 4^2}{12} \end{Bmatrix}$$

$$= \begin{Bmatrix} -4.487 \\ 0 \\ -108.97 \end{Bmatrix}$$



P2



$$K_{II} = K_{22}(1, 1)$$

$$K_{22} = \frac{EA}{L} \left( \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} (1, 0) + \begin{bmatrix} -1/2 \\ -\sqrt{3}/2 \end{bmatrix} \begin{bmatrix} -\frac{1}{2} & -\frac{\sqrt{3}}{2} \end{bmatrix} \right) = \frac{EA}{L} \begin{pmatrix} 5/4 & \sqrt{3}/4 \\ \sqrt{3}/4 & 3/4 \end{pmatrix}$$

$$K_{II} = \frac{EA}{L} \left( \frac{5}{4} \right) = 9 \cdot \frac{2}{3} \cdot \frac{1}{3} \cdot \frac{5}{4} = \frac{5}{2}$$

$$P_{I, \text{mech}} = P_a = -P \frac{\sqrt{3}}{2} = -\frac{\sqrt{3}}{2}$$

$$\begin{aligned} P_{2, \text{temp+settle}} &= a_{23}^T \left[ \frac{EA}{L} (-\alpha \Delta T L) + \frac{EA}{L} a_{32} \begin{Bmatrix} 0 \\ S \end{Bmatrix} \right] \text{ settlement} \\ &= 9 \cdot \frac{2}{3} \cdot \frac{1}{3} \begin{bmatrix} -1/2 \\ -\sqrt{3}/2 \end{bmatrix} \begin{bmatrix} -0.01 \\ 20.3 \end{bmatrix} \begin{bmatrix} \sqrt{3}/2 & 1/2 \end{bmatrix} \begin{Bmatrix} 0 \\ 0.2 \end{Bmatrix} \\ &= \begin{bmatrix} 0.5 \\ \sqrt{3}/2 \end{bmatrix} \tilde{P}^e \end{aligned}$$

$$P_I = P_a - \tilde{P}^e = -\frac{\sqrt{3}}{2} - \frac{1}{2} = -1.366 \text{ kN}$$

$$\Delta_I = K_{II}^{-1} P_I = -\frac{2}{5} (1.366) = -0.5464$$

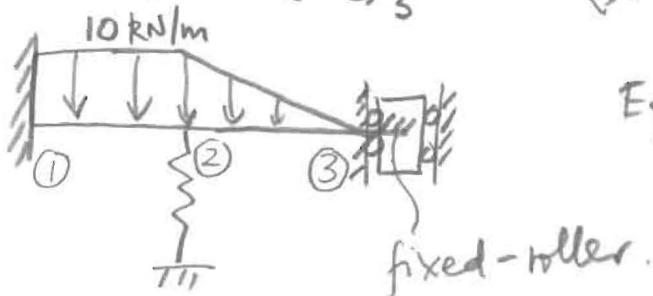
$$F_{21} = \frac{EA}{L} a_{21} \Delta_2 = 2 [1 \ 0] \begin{bmatrix} -0.5464 \\ 0 \end{bmatrix} = -1.0928 \text{ kN}$$

(ie, Compressive)

$$\begin{aligned} F_{23} &= \frac{EA}{L} a_{23} \Delta_2 + F_{s23}^f = 2 \begin{bmatrix} -1/2 & -\sqrt{3}/2 \end{bmatrix} \begin{bmatrix} -0.5464 \\ 0 \end{bmatrix} + [-1] \\ &= -0.4536 \text{ (compressive)} \end{aligned}$$

(2)

P3 Symmetry  $\Rightarrow (\Delta_6)_3 = 0$  ( $\text{re}(\Theta)_3 = 0$ ). (3)



Equivalent half-structure

Can use beam formulation ( $\because$  no horizontal loads)

$$K_{II} = \text{keep rows/cols } (3, 4, 5) = \begin{bmatrix} K_{22}(1,1) + k & K_{22}(1,2) & K_{23}(1,1) \\ K_{22}(2,1) & K_{22}(2,2) & K_{23}(2,1) \\ K_{32}(1,1) & K_{32}(1,2) & K_{33}(1,1) \end{bmatrix}$$

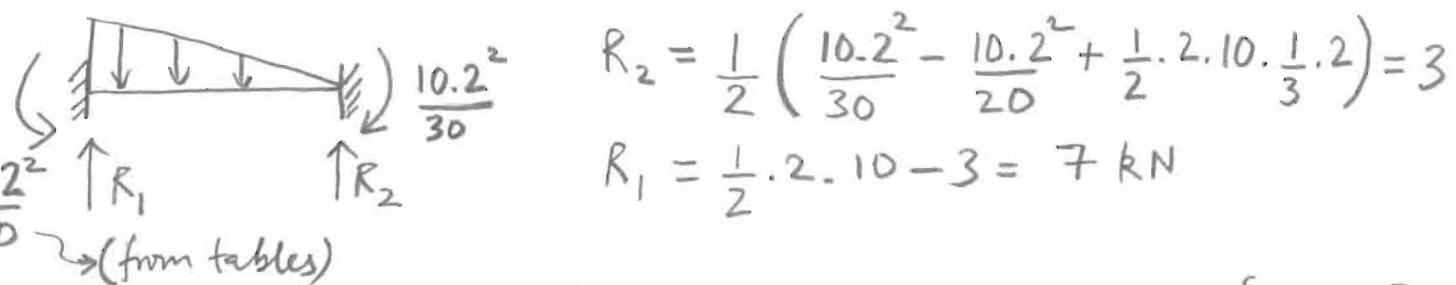
$$a_{21} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} = a_{32}, \quad a_{23} = \begin{bmatrix} -1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$K_{22} = \frac{EI}{L} \left( \begin{bmatrix} 12/L^2 & -6/L \\ -6/L & 4 \end{bmatrix} + \begin{bmatrix} 12/L^2 & 6/L \\ 6/L & 4 \end{bmatrix} \right) = \frac{2EI}{L} \begin{bmatrix} 12/L^2 & 0 \\ 0 & 4 \end{bmatrix}$$

$$K_{33} = k^2 = \frac{EI}{L} \begin{bmatrix} 12/L^2 & -6/L \\ -6/L & 4 \end{bmatrix}$$

$$K_{23} = \frac{EI}{L} \begin{bmatrix} -12/L^2 & 6/L \\ -6/L & 2 \end{bmatrix}, \quad K_{32} = K_{23}^T$$

$$K_{II} = \frac{EI}{L} \begin{bmatrix} 24/L^2 + \frac{L}{EI}k & 0 & -12/L^2 \\ 0 & 8 & -6/L \\ -12/L^2 & -6/L & 12/L^2 \end{bmatrix} = 5000 \begin{bmatrix} 7 & 0 & -3 \\ 0 & 8 & -3 \\ -3 & -3 & 3 \end{bmatrix}$$



$$R_2 = \frac{1}{2} \left( \frac{10.2^2}{30} - \frac{10.2^2}{20} + \frac{1}{2} \cdot 2 \cdot 10 \cdot \frac{1}{3} \cdot 2 \right) = 3$$

$$R_1 = \frac{1}{2} \cdot 2 \cdot 10 - 3 = 7 \text{ kN}$$

$\frac{10.2^2}{20} \rightarrow$  (from tables)

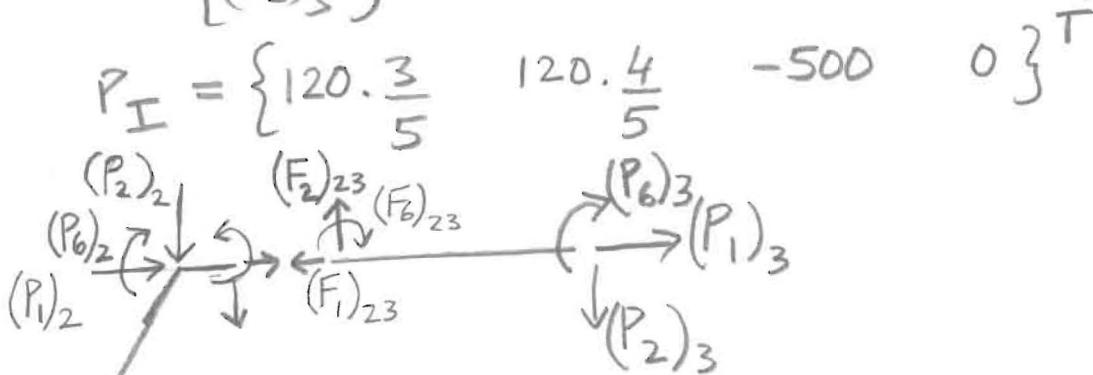
$$P_I = \begin{Bmatrix} \frac{10.2}{2} + 7 \\ -\frac{10.2^2}{12} + \frac{10.2^2}{20} \\ 3 \end{Bmatrix} = \begin{Bmatrix} 17 \\ -4/3 \\ 3 \end{Bmatrix} ; \quad \Delta_I = K_{II}^{-1} P_I = \begin{Bmatrix} 1.909 \\ 1.212 \\ 3.321 \end{Bmatrix} \cdot 10^{-3}$$

↓③ ←  
G④ ←

(4)

P4 Reactions at node ③ are redundant.  
Only external redundants.

$$P_{II} = \begin{Bmatrix} (P_1)_3 \\ (P_2)_3 \end{Bmatrix} = D_{III}^{-1} (-D_{II} P_I)$$



$$\begin{Bmatrix} (F_2)_{12} \\ (F_1)_{12} \end{Bmatrix} = \begin{Bmatrix} (F_1)_{12} \\ (F_2)_{12} \\ (F_6)_{12} \\ (F_1)_{23} \\ (F_2)_{23} \\ (F_6)_{23} \end{Bmatrix} = \left[ \begin{array}{cccc|cc} 0.6 & -0.8 & 0 & 0 & 0.6 & -0.8 \\ 0.8 & 0.6 & 0 & 0 & 0.8 & 0.6 \\ -16 & -12 & -1 & -1 & -16 & -32 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & -1 & 0 & -20 \end{array} \right] \begin{Bmatrix} (P_1)_2 \\ (P_2)_2 \\ (P_6)_2 \\ (P_1)_3 \\ (P_6)_3 \\ (P_2)_3 \end{Bmatrix}$$

$$\beta = \frac{EI}{EA}, L_{12} = L_{23} = L$$

$$du = \frac{L}{EI} \begin{bmatrix} B & 0 & 0 \\ 0 & L^2/3 & L/2 \\ 0 & L/2 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} = \frac{20}{EI} \begin{bmatrix} 0.01 & 0 & 0 \\ 0 & 400/3 & 10 \\ 0 & 10 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$$

$$b_{II}^T du = \begin{bmatrix} 0.006 & -160/3 & -8 & 0.01 & 0 & 0 \\ -0.008 & -240 & -26 & 0 & -200/3 & -10 \end{bmatrix}$$

$$D_{II} = b_{II}^T du b_{II} = \begin{bmatrix} 85.347 & 223.995 \\ 223.995 & 821.340 \end{bmatrix} \cdot \frac{L}{EI} \cdot 20$$

$$D_{II} = b_{II}^T du b_{II} = \begin{bmatrix} 85.337 & 63.995 & 8 & 8 \\ 223.995 & 168.006 & 26 & 36 \end{bmatrix} \cdot \frac{L}{EI} \cdot 20$$

$$D_{II\|} P_I = \{8287.784 \quad 19256.216\}^T \cdot \frac{L}{EI} \cdot 20$$

(5)

$$D_{II\|}^{-1} = \begin{bmatrix} 0.0412 & -0.0112 \\ -0.0112 & 0.0043 \end{bmatrix} \cdot \frac{EI}{L} \cdot \frac{1}{20}$$

$$P_{II} = \{-125.158\}$$

