

DEPARTMENT OF CIVIL ENGINEERING
CE-317 STRUCTURAL MECHANICS II
 Midsem 11/9/13

PAPER CODE: A

Note: Write your name and roll no. on the answerbook and on the summary answer sheet provided on the reverse.

You must submit the question-paper-cum-summary-answer-sheet along with the answerbook.

Closed book, closed notes test. No formula sheet allowed. No mobile phones allowed in the exam hall.

All **three questions** carry equal marks

Problem 1

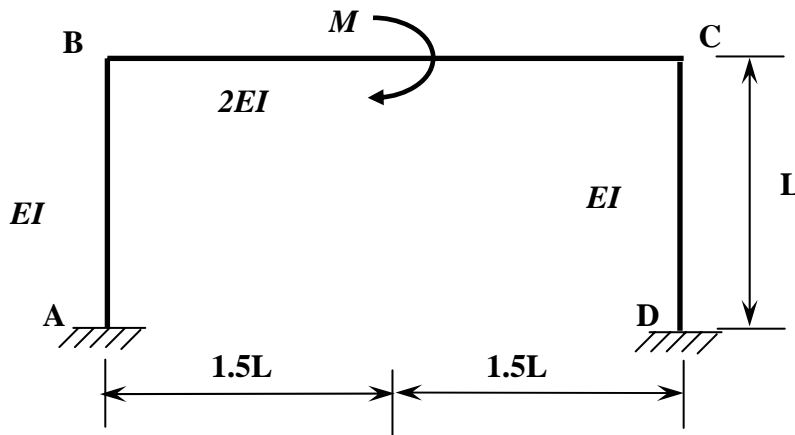
You must use only Slope Deflection Method in this question.

Consider the rigid-jointed frame in **Fig. 1**. A point moment M is applied at the midpoint of member BC as shown. **Find:**

- (i) **Rotation at B.**
- (ii) **All reactions at A.**

(Fixed end moment due to point moment M applied at mid-span is $M/4$)

Fig. 1



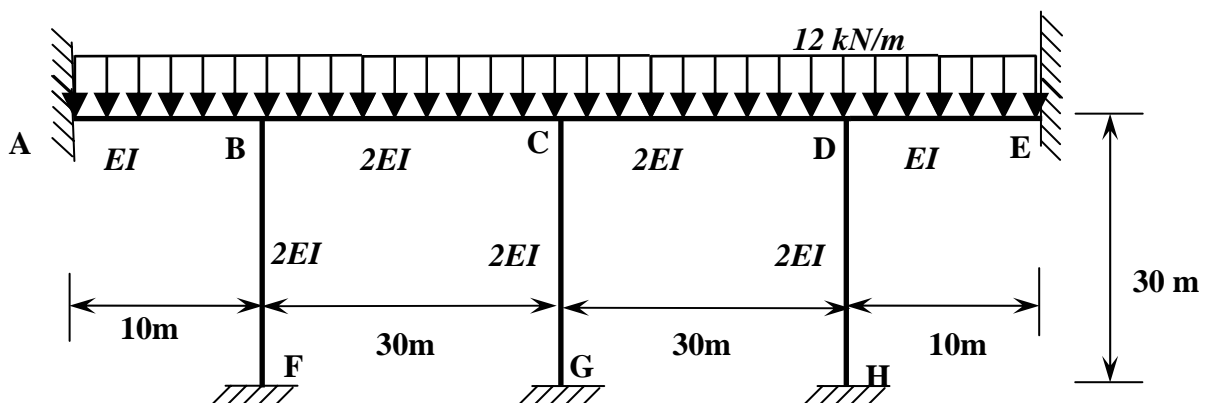
Problem 2

You must use only Moment Distribution Method in this question.

For the rigid-jointed frame in **Fig. 2**, find all reactions at F

(Fixed end moment due to uniformly distributed load is $wL^2/12$)

Fig. 2



Problem 3

You must use only Stiffness Method in this question. (You must use joint numbering and global coordinate system as shown in Fig. 3. Assume unit of EA is kN.)

For the 4-member pin-jointed truss with load as shown in Fig. 3, find:

- (i) The \mathbf{K}_{II} (i.e., the reduced stiffness matrix \mathbf{K} -roman-one-roman-one).
- (ii) Displacement of joints 2 and 3.

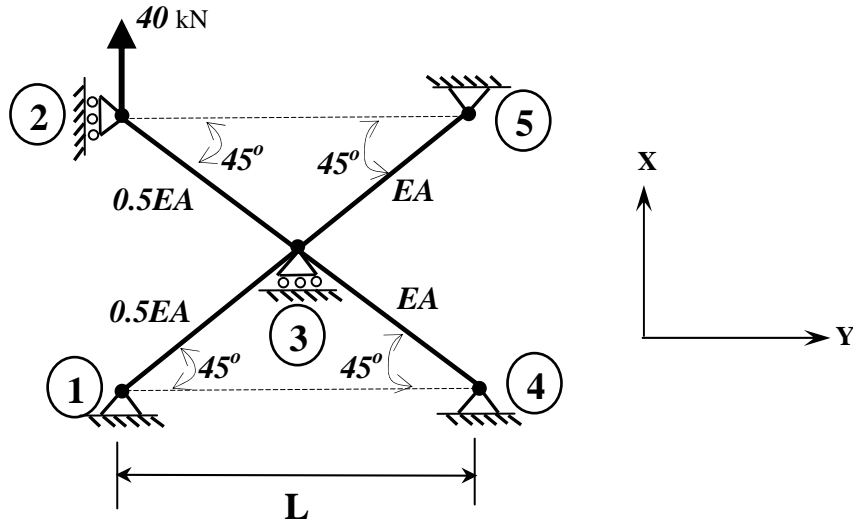


Fig. 3

SUMMARY ANSWER SHEET

PAPER CODE: A

Name:

Roll no:

Problem 1

- (i) Rotation at $B =$
- (ii) Reactions at A are $A_x =$; $A_y =$; $M_A =$

Problem 2

Reactions at F are $F_x =$; $F_y =$; $M_F =$

Problem 3

- (i) Reduced stiffness matrix $\mathbf{K}_{II} =$
- (ii) Displacement of joint 2 is =
Displacement of joint 3 is =

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PAPER CODE: B

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Problem 1

You must use only Slope Deflection Method in this question.

Consider the rigid-jointed frame in **Fig. 1**. A point moment M is applied at the midpoint of member BC as shown. **Find:**

- (iii) **Rotation at B .**
- (iv) **All reactions at A .**

(Fixed end moment due to point moment M applied at mid-span is $M/4$)

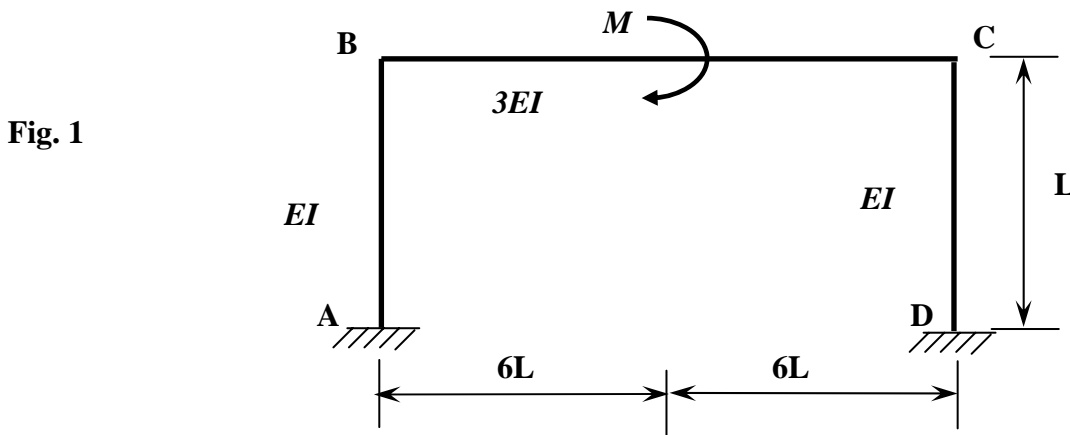


Fig. 1

Problem 2

You must use only Moment Distribution Method in this question.

For the rigid-jointed frame in **Fig. 2**, find all reactions at F

(Fixed end moment due to uniformly distributed load is $wL^2/12$)

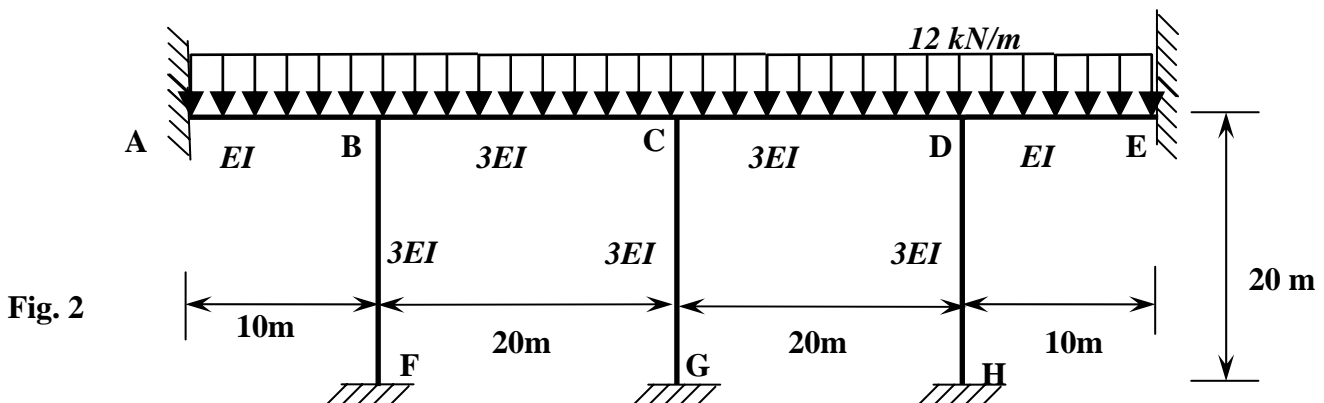


Fig. 2

Problem 3

You must use only Stiffness Method in this question. (You must use joint numbering and global coordinate system as shown in Fig. 3. Assume unit of EA is kN.)

For the 4-member pin-jointed truss with load as shown in Fig. 3, find:

- (i) The K_{II} (i.e., the reduced stiffness matrix K -roman-one-roman-one).
- (ii) Displacement of joints 2 and 3.

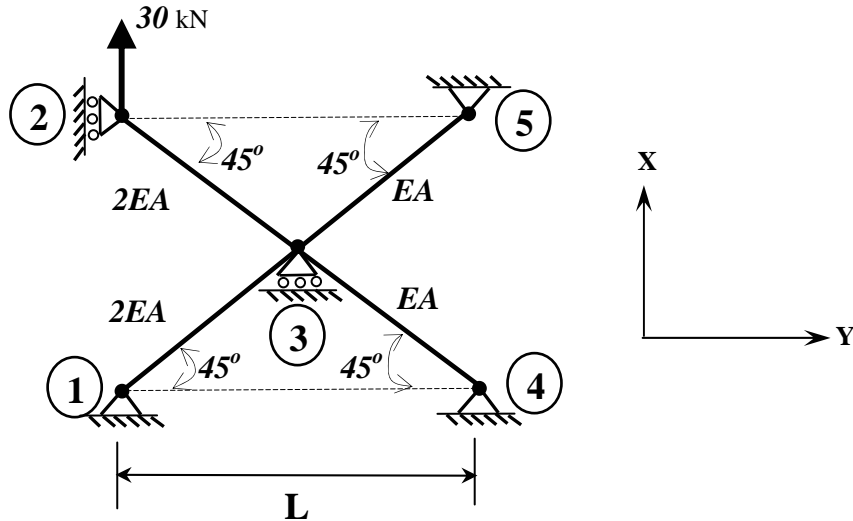


Fig. 3

SUMMARY ANSWER SHEET

PAPER CODE: B

Name:

Roll no:

Problem 1

(iii) Rotation at B =

(iv) Reactions at A are $A_x =$; $A_y =$; $M_A =$

Problem 2

Reactions at F are $F_x =$; $F_y =$; $M_F =$

Problem 3

(iii) Reduced stiffness matrix $K_{II} =$

(iv) Displacement of joint 2 is =

Displacement of joint 3 is =

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PAPER CODE: C

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All **three questions** carry equal marks

Problem 1

You must use only Slope Deflection Method in this question.

Consider the rigid-jointed frame in **Fig. 1**. A point moment M is applied at the midpoint of member BC as shown. **Find:**

- (i) **Rotation at B.**
- (ii) **All reactions at A.**

(Fixed end moment due to point moment M applied at mid-span is $M/4$)

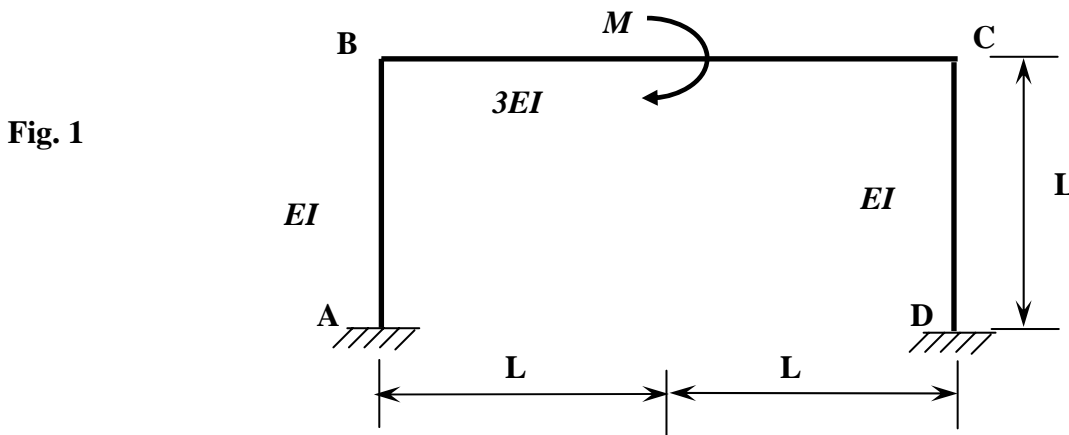


Fig. 1

Problem 2

You must use only Moment Distribution Method in this question.

For the rigid-jointed frame in **Fig. 2**, find all reactions at F

(Fixed end moment due to uniformly distributed load is $wL^2/12$)

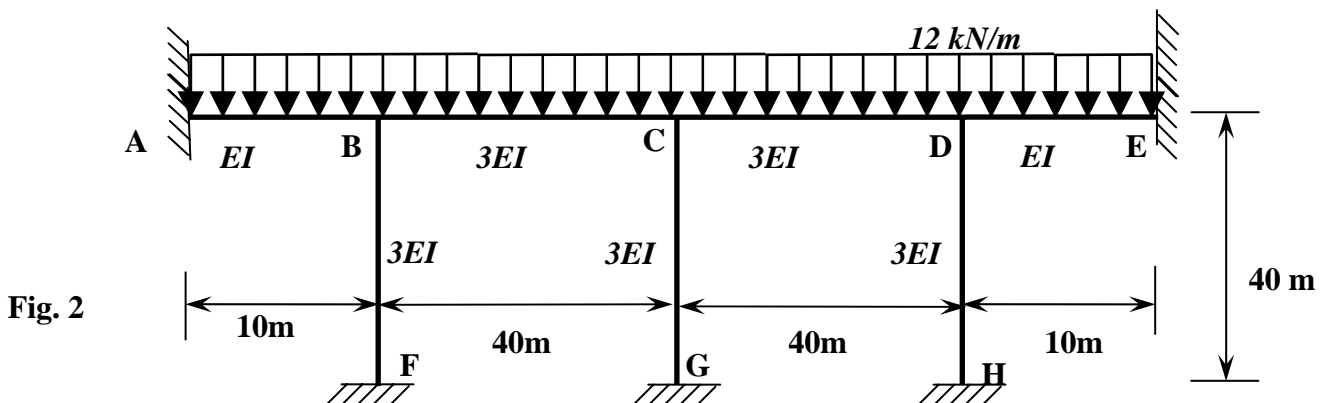


Fig. 2

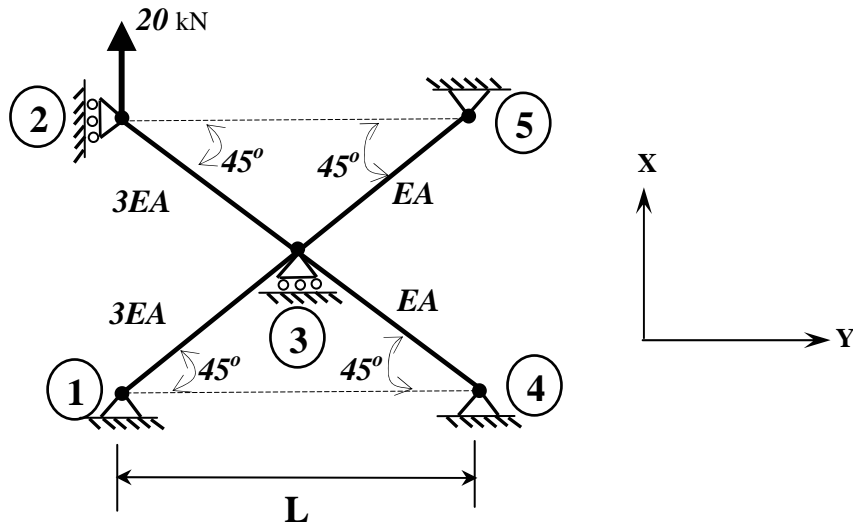
Problem 3

You must use only Stiffness Method in this question. (You must use joint numbering and global coordinate system as shown in Fig. 3. Assume unit of EA is kN.)

For the 4-member pin-jointed truss with load as shown in Fig. 3, find:

- (i) The \mathbf{K}_{II} (i.e., the reduced stiffness matrix \mathbf{K} -roman-one-roman-one).
- (ii) Displacement of joints 2 and 3.

Fig. 3



SUMMARY ANSWER SHEET

PAPER CODE: C

Name:

Roll no:

Problem 1

(i) Rotation at $B =$

(ii) Reactions at A are $A_x =$; $A_y =$; $M_A =$

Problem 2

Reactions at F are $F_x =$; $F_y =$; $M_F =$

Problem 3

(i) Reduced stiffness matrix $\mathbf{K}_{II} =$

(ii) Displacement of joint 2 is =

Displacement of joint 3 is =

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PAPER CODE: D

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 Closed book, closed notes test. No formula sheet allowed. No mobile phones allowed in the exam hall.
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Problem 1

You must use only Slope Deflection Method in this question.

Consider the rigid-jointed frame in **Fig. 1**. A point moment M is applied at the midpoint of member BC as shown. **Find:**

- (iii) **Rotation at B.**
- (iv) **All reactions at A.**

(Fixed end moment due to point moment M applied at mid-span is $M/4$)

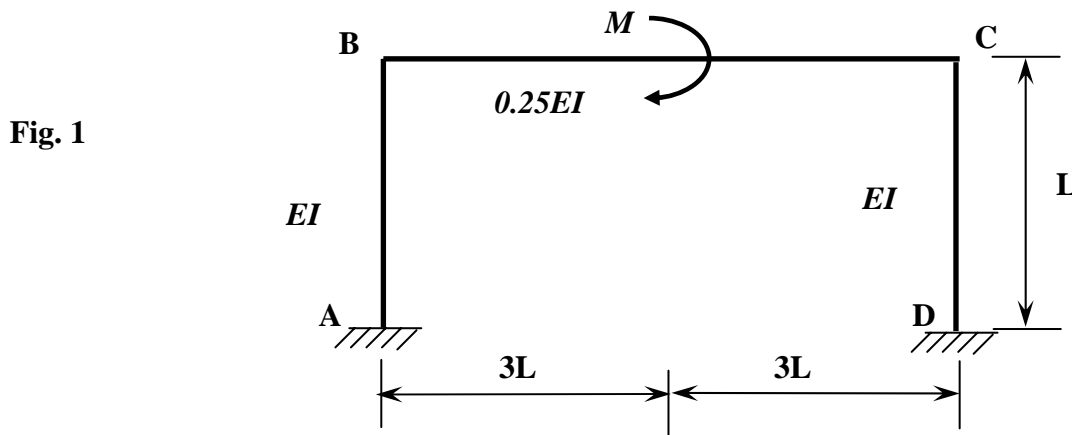


Fig. 1

Problem 2

You must use only Moment Distribution Method in this question.

For the rigid-jointed frame in **Fig. 2**, find all reactions at F

(Fixed end moment due to uniformly distributed load is $wL^2/12$)

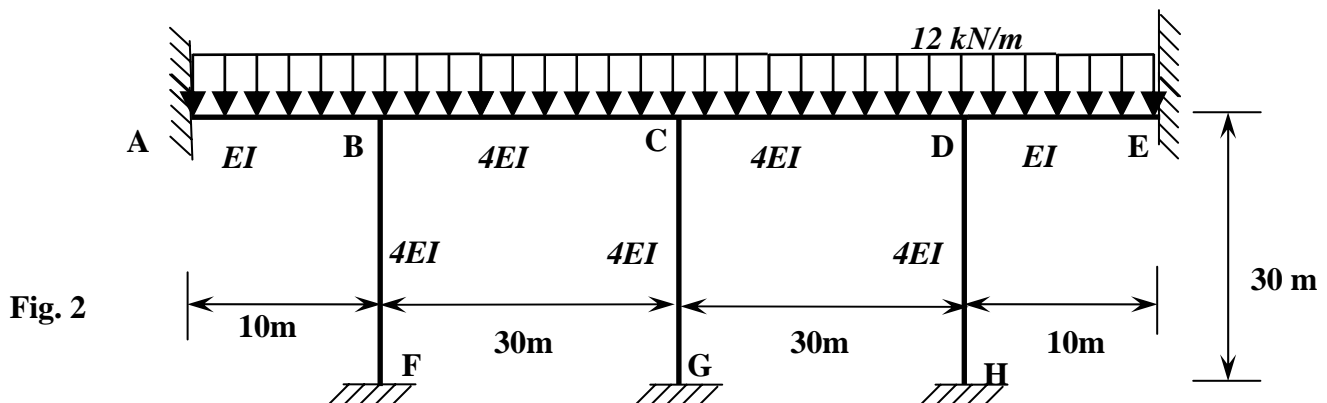


Fig. 2

Problem 3

You must use only Stiffness Method in this question. (You must use joint numbering and global coordinate system as shown in Fig. 3. Assume unit of EA is kN.)

For the 4-member pin-jointed truss with load as shown in Fig. 3, find:

- (i) The \mathbf{K}_{II} (i.e., the reduced stiffness matrix \mathbf{K} -roman-one-roman-one).
- (ii) Displacement of joints 2 and 3.

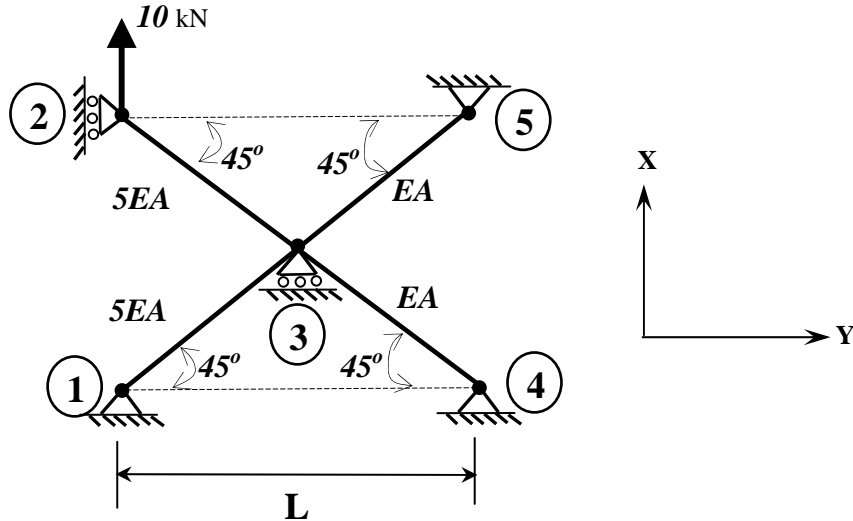


Fig. 3

SUMMARY ANSWER SHEET

PAPER CODE: D

Name:

Roll no:

Problem 1

(iii) Rotation at $B =$

(iv) Reactions at A are $A_x =$; $A_y =$; $M_A =$

Problem 2

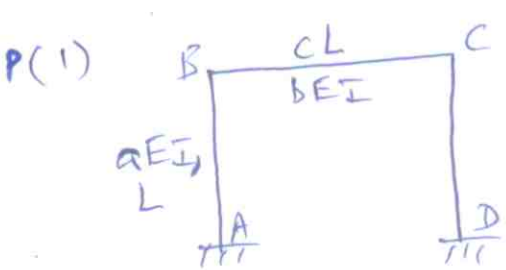
Reactions at F are $F_x =$; $F_y =$; $M_F =$

Problem 3

(iii) Reduced stiffness matrix $\mathbf{K}_{II} =$

(iv) Displacement of joint 2 is =

Displacement of joint 3 is =



Antisymm $\Rightarrow \theta_B = \theta_C$
 \hookrightarrow sway.

$$M_{BA} + M_{BC} = 0 \rightarrow \frac{EI}{L} \left[\frac{a}{1} \cdot 4 + \frac{b}{c} \cdot 4 \right] \theta_B - \frac{6}{L} \cdot \frac{a}{1} \Delta = -\frac{M}{4}$$

$$M_{CB} + M_{CD} = 0 \rightarrow \frac{EI}{L} \left[\frac{a}{1} \cdot 2 + \frac{a}{1} \cdot 4 \right] \theta_C - \frac{6}{L} \cdot \frac{a}{1} \Delta = -\frac{M}{4}$$

$$V_A + V_D = 0 \rightarrow \frac{EI}{L} \left[\frac{a}{1} \cdot 2 + \frac{a}{1} \cdot 4 \right] \Delta = 0$$

Not required \therefore using antisymmetry.

$$\text{use } \theta_B = \theta_C \rightarrow \left. \begin{aligned} (4a + 6\frac{b}{c})\theta_B - \frac{6}{L}a\Delta &= -\frac{M}{4} \cdot \frac{L}{EI} \\ 12a\theta_B - \frac{24}{L}a\Delta &= 0 \end{aligned} \right\}$$

$$\Rightarrow (4a + 6\frac{b}{c} - \frac{6}{2}a)\theta_B = -\frac{M}{4} \frac{L}{EI}$$

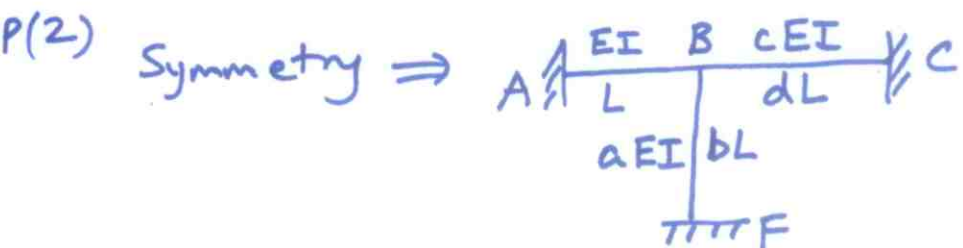
$$\theta_B = -\frac{M}{4} \cdot \frac{1}{(a + 6b/c)} \cdot \frac{L}{EI}$$

$$\Delta = -\frac{M}{4} \frac{1}{(a + 6b/c)} \frac{L}{EI} \cdot \frac{L}{2}$$

$$M_{AB} = a \frac{EI}{L} \left(2 \left(-\frac{M}{4} \right) \frac{1}{(a + 6b/c)} - 6 \left(-\frac{M}{4} \right) \frac{1}{(a + 6b/c)} \cdot \frac{L}{2} \cdot \frac{1}{L} \right) \cdot \frac{L}{EI}$$

$$\frac{R/A}{A_y} = \frac{(2M_{AB} + M)}{CL} ; \therefore M_{AB} = -M_{BA}, A_x = 0$$

Paper code	a	b	c	θ_B	M_{AB}	A_y	A_x
A	1	2	3	$-\frac{ML}{EI} \cdot \frac{1}{20}$	$M \cdot \frac{1}{20}$	$\frac{M}{L} \cdot \frac{11}{30}$	0
B	1	3	12	$-\frac{ML}{EI} \cdot \frac{1}{10}$	$M \cdot \frac{1}{10}$	$\frac{M}{L} \cdot \frac{1}{10}$	0
C	1	3	2	$-\frac{ML}{EI} \cdot \frac{1}{40}$	$M \cdot \frac{1}{40}$	$\frac{M}{L} \cdot \frac{21}{40}$	0
D	1	0.25	6	$-\frac{ML}{EI} \cdot \frac{1}{5}$	$M \cdot \frac{1}{5}$	$\frac{M}{L} \cdot \frac{7}{30}$	0



mem end	AB	BA	BF	BC	CB	FB
rel stiff	1	1	a/b	c/d	c/d	a/b
df	—	$\frac{1}{1 + \frac{a}{b} + \frac{c}{d}}$	$\frac{a/b}{1 + \frac{a}{b} + \frac{c}{d}}$	$\frac{c/d}{1 + \frac{a}{b} + \frac{c}{d}}$	—	—
fem	$-\frac{wL^2}{12}$	$\frac{wL^2}{12}$	0	$-\frac{wd^2L^2}{12}$	$\frac{wd^2L^2}{12}$	0
df, co	q/2	q	r	s	s/2	r/2
BM	$\frac{q}{2} - \frac{wL^2}{12}$	$q + \frac{wL^2}{12}$	r	$s - \frac{wd^2L^2}{12}$	$\frac{s}{2} + \frac{wd^2L^2}{12}$	r/2

where,

$$q = \frac{wL^2}{12} (d^2 - 1) \cdot \frac{bd}{bd + ad + bc} = \frac{24}{7} \cdot 100$$

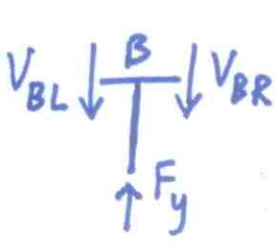
$$r = \frac{wL^2}{12} (d^2 - 1) \cdot \frac{ad}{bd + ad + bc} = \frac{16}{7} \cdot 100$$

$$s = \frac{wL^2}{12} (d^2 - 1) \cdot \frac{bc}{bd + ad + bc} = \frac{16}{7} \cdot 100$$

for Paper Code A.

BM	71.43	442.86	228.57	-671.43	1014.29	114.29
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$$M_F = M_{FB} = 114.29 ; F_x = \frac{M_{FB} + M_{BF}}{BL} = \frac{114.29 + 228.57}{30} = 11.43$$



$$F_y = V_{BL} + V_{BR} = \frac{M_{AB} + M_{BA} + wL^2/2}{L} + \frac{(-M_{BC} - M_{CB} + wd^2/2)}{dL}$$

$$= (71.43 + 442.86 + 600)/10 + \frac{1}{30} (+671.43 - 1014.29 + 5400)$$

$$= 280$$

Paper Code	M_F	F_x	F_y
A	114.29	11.43	280
B	56.25	8.44	182.81
C	225	16.88	373.13
D	145.45	14.54	258.18

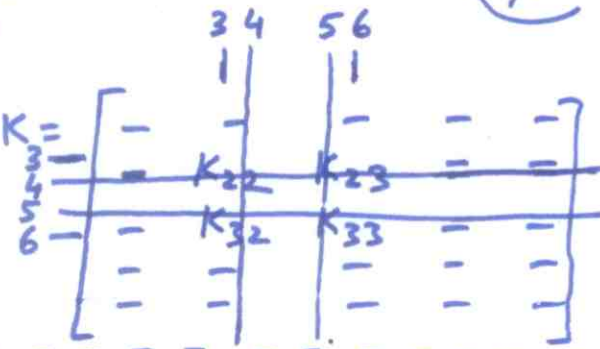
(for MDM tables)
(see xls file)

Problem2	Paper A										
	ab	ba	bf	bc	cb	fb		aEI	bL	cEI	dL
k	1	1	0.666667	0.666667	0.666667	0.666667		2	3	2	3
df		0.428571	0.285714	0.285714				L	w		
fem	-100	100	0	-900	900	0		10	12		
dist, co	171.4286	342.8571	228.5714	228.5714	114.2857	114.2857		Mf	Fx	VbL	VbR
BM	71.42857	442.8571	228.5714	-671.429	1014.286	114.2857		114.2857	11.42857	111.4286	168.5714
									Fy		
									280		
Problem2	Paper B										
	ab	ba	bf	bc	cb	fb		aEI	bL	cEI	dL
k	1	1	1.5	1.5	1.5	1.5		3	2	3	2
df		0.25	0.375	0.375				L	w		
fem	-100	100	0	-400	400	0		10	12		
dist, co	37.5	75	112.5	112.5	56.25	56.25		Mf	Fx	VbL	VbR
BM	-62.5	175	112.5	-287.5	456.25	56.25		56.25	8.4375	71.25	111.5625
									Fy		
									182.8125		
Problem2	Paper C										
	ab	ba	bf	bc	cb	fb		aEI	bL	cEI	dL
k	1	1	0.75	0.75	0.75	0.75		3	4	3	4
df		0.4	0.3	0.3				L	w		
fem	-100	100	0	-1600	1600	0		10	12		
dist, co	300	600	450	450	225	225		Mf	Fx	VbL	VbR
BM	200	700	450	-1150	1825	225		225	16.875	150	223.125
									Fy		
									373.125		
Problem2	Paper D										
	ab	ba	bf	bc	cb	fb		aEI	bL	cEI	dL
k	1	1	1.333333	1.333333	1.333333	1.333333		4	3	4	3
df		0.272727	0.363636	0.363636				L	w		
fem	-100	100	0	-900	900	0		10	12		
dist, co	109.0909	218.1818	290.9091	290.9091	145.4545	145.4545		Mf	Fx	VbL	VbR
BM	9.090909	318.1818	290.9091	-609.091	1045.455	145.4545		145.4545	14.54545	92.72727	165.4545
									Fy		
									258.1818		

$$(P3) \Delta_I = \{(\Delta_2)_2 (\Delta_1)_3\}^T = \{\Delta_4 \Delta_5\}^T$$

K_{II} = keep cols/rows 4,5 of $K \rightarrow$

$$\Rightarrow K_{II} = \begin{bmatrix} K_{22}(2,2) & K_{23}(2,1) \\ K_{32}(1,2) & K_{33}(1,1) \end{bmatrix}$$



$$a_{31} = \left(\frac{1}{\sqrt{2}} \quad \frac{1}{\sqrt{2}}\right) = -a_{35}$$

$$a_{32} = \left(\frac{1}{\sqrt{2}} \quad -\frac{1}{\sqrt{2}}\right) = -a_{23} = -a_{34}$$

$$\begin{aligned} R_{23} = R_{32} = R_{22}^3 = R_{33}^2 = R_{13} = R_{31} \\ = R_{11}^3 = R_{33}^1 = aEA/(L/\sqrt{2}) \\ R_{34} = R_{43} = R_{35} = R_{53} = R_{33}^5 = R_{55}^3 = R_{33}^4 \\ = R_{44}^5 = EA/(L/\sqrt{2}) \end{aligned}$$

$$K_{22} = a_{23}^T R_{22}^3 a_{23} = \frac{EA}{L} \cdot \frac{a}{1/\sqrt{2}} \begin{pmatrix} -1/\sqrt{2} \\ 1/\sqrt{2} \end{pmatrix} \begin{pmatrix} -1/\sqrt{2} & 1/\sqrt{2} \end{pmatrix} = \frac{EA}{L} \begin{bmatrix} X & X \\ X & \frac{a\sqrt{2}}{2} \end{bmatrix}$$

where axial stiffness of members 1-3, 2-3 is $a \cdot EA$

$$K_{33} = a_{32}^T R_{33}^2 a_{32} + a_{31}^T R_{33}^1 a_{31} + a_{34}^T R_{33}^4 a_{34} + a_{35}^T R_{33}^5 a_{35}$$

$$= \frac{EA}{L} \left\{ (\sqrt{2} + a\sqrt{2}) \begin{pmatrix} 1/\sqrt{2} \\ 1/\sqrt{2} \end{pmatrix} \begin{pmatrix} 1/\sqrt{2} & 1/\sqrt{2} \end{pmatrix} + (\sqrt{2} + a\sqrt{2}) \begin{pmatrix} 1/\sqrt{2} \\ -1/\sqrt{2} \end{pmatrix} \begin{pmatrix} 1/\sqrt{2} & -1/\sqrt{2} \end{pmatrix} \right\}$$

$$= \frac{EA}{L} \begin{bmatrix} \frac{\sqrt{2}}{2}(a+1) \cdot 2 & X \\ X & X \end{bmatrix}$$

$$K_{23} = \frac{EA}{L} a\sqrt{2} \begin{pmatrix} -1/\sqrt{2} \\ 1/\sqrt{2} \end{pmatrix} \begin{pmatrix} 1/\sqrt{2} & -1/\sqrt{2} \end{pmatrix} = \frac{EA}{L} a\sqrt{2} \begin{bmatrix} -1/2 & 1/2 \\ 1/2 & -1/2 \end{bmatrix} = K_{32}$$

$$= a_{23}^T R_{23} a_{32}$$

$$K_{II} = \frac{EA}{L} \begin{bmatrix} a/\sqrt{2} & a/\sqrt{2} \\ a/\sqrt{2} & (a+1)\sqrt{2} \end{bmatrix} ; K_{II}^{-1} = \left(a^2 + a - \frac{a^2}{2}\right)^{-1} \frac{L}{EA} \begin{bmatrix} (a+1)\sqrt{2} - a/\sqrt{2} \\ -a/\sqrt{2} & a/\sqrt{2} \end{bmatrix}$$

$$\Delta_I = K_{II}^{-1} \begin{Bmatrix} P_4 \\ P_5 \end{Bmatrix} = \left(a + \frac{a^2}{2}\right)^{-1} \frac{L}{EA} \begin{Bmatrix} \sqrt{2}(a+1)P_4 - \frac{a}{\sqrt{2}}P_5 \\ -\frac{a}{\sqrt{2}}P_4 + \frac{a}{\sqrt{2}}P_5 \end{Bmatrix} = \begin{Bmatrix} \Delta_4 \\ \Delta_5 \end{Bmatrix}$$

$P_4 = a$ as given, $P_5 = 0$, $a = a$ as given

Paper code	P_4	P_5	a	$\Delta_4 = (\Delta_2)_2$	$\Delta_5 = (\Delta_1)_3$
A	40	0	0.5	$135.76 L/EA$	$-22.63 L/EA$
B	30	0	2	$31.82 L/EA$	$-10.61 L/EA$
C	20	0	3	$15.08 L/EA$	$-5.66 L/EA$
D	10	0	5	$4.85 L/EA$	$-2.02 L/EA$