

→ The aim is to eliminate all v_i 's and e_i 's in favor of v & e .

P-1.

$e = e_3 = e_1 + e_2 \rightarrow (d)$		$v_1 = k_1 e_1 \rightarrow (e)$
$e_1 = e_4 \rightarrow (b)$		$v_2 = c_2 \dot{e}_2 \rightarrow (f)$
$v_2 = v_1 + v_4 \rightarrow (c)$		$v_3 = k_3 e_3 \rightarrow (g)$
$v = v_3 + v_2 \rightarrow (d)$		$v_4 = c_4 \dot{e}_4 \rightarrow (h)$

Using (d, f, g, a₁, a₂)

$$v = v_3 + v_2 = k_3 e_3 + c_2 \dot{e}_2 = k_3 e + c_2 (\dot{e} - \dot{e}_1) \rightarrow (1)$$

Using (a₂, b, f, h, c, d) g, a₁.

Also, $\dot{e} = \dot{e}_1 + \dot{e}_2 = \frac{v_4}{c_4} + \frac{v_2}{c_2} = \frac{v_2 - v_1}{c_4} + \frac{v - v_3}{c_2}$

$$= \frac{v - v_3 - v_1}{c_4} + \frac{v - v_3}{c_2} = (v - \underbrace{v_3}_{=k_3 e}) \left(\frac{1}{c_4} + \frac{1}{c_2} \right) - \frac{v_1}{c_4}$$

$$\dot{e} = (v - k_3 e) \left(\frac{1}{c_4} + \frac{1}{c_2} \right) - \frac{v_1}{c_4} \rightarrow (2)$$

Substitute (2), (e) in (1):

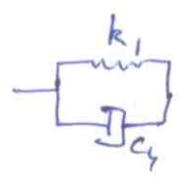
$$v = k_3 e + c_2 \left(\dot{e} - \frac{v_1}{c_4} \right) = k_3 e + c_2 \left(\dot{e} - \frac{1}{k_1} \left\{ c_4 \left[(v - k_3 e) \left(\frac{c_2 + c_4}{c_2 c_4} \right) - \dot{e} \right] \right\} \right)$$

$$= k_3 e + c_2 \left(\dot{e} + \frac{c_4}{k_1} \ddot{e} - \frac{c_4}{k_1} \left(\frac{c_2 + c_4}{c_2 c_4} \right) \dot{v} + \frac{c_4}{k_1} k_3 \left(\frac{c_2 + c_4}{c_2 c_4} \right) \dot{e} \right)$$

$$v + \left(\frac{c_2 + c_4}{k_1} \right) \dot{v} = k_3 e + \left[c_2 + \frac{k_3}{k_1} (c_2 + c_4) \right] \dot{e} + \left(\frac{c_2 c_4}{k_1} \right) \ddot{e}$$

Alternative - I

Use Voigt element



We have, $v_v = v_1 + v_4$, $e_v = e_1 = e_4$, $v_1 = k_1 e_1$, $v_4 = c_4 \dot{e}_4$

$$\Rightarrow v_v = k_1 e_v + c_4 \dot{e}_v$$

Also, $v_v = v_2$, $e = e_3 = e_v + e_2$, $v = v_2 + v_3$, $v_2 = c_2 \dot{e}_2$, $v_3 = k_3 e_3$

$$\begin{aligned}
 \Rightarrow \dot{v} &= \dot{v}_2 + \dot{v}_3 = c_2 \ddot{e}_2 + k_3 \dot{e}_3 = c_2 (\ddot{e} - \ddot{e}_v) + k_3 \dot{e} \\
 &= c_2 \left(\ddot{e} - \frac{1}{c_4} [\dot{v}_v - k_1 \dot{e}_v] \right) + k_3 \dot{e} = \\
 &= c_2 \left(\ddot{e} - \frac{1}{c_4} [\dot{v}_2 - k_1 (\dot{e} - \dot{e}_2)] \right) + k_3 \dot{e} = c_2 \left(\ddot{e} - \frac{1}{c_4} [\dot{v} - \dot{v}_3 - k_1 (\dot{e} - \frac{\sigma}{c_2})] \right) + k_3 \dot{e} \\
 &= c_2 \left(\ddot{e} - \frac{1}{c_4} [\dot{v} - k_3 \dot{e}_3 - k_1 (\dot{e} - \frac{\sigma - \sqrt{3}}{c_2})] \right) + k_3 \dot{e} \\
 &= c_2 \left(\ddot{e} - \frac{1}{c_4} [\dot{v} - k_3 \dot{e} - k_1 (\dot{e} - \frac{\sigma - k_3 e_3}{c_2})] \right) + k_3 \dot{e} \\
 &= c_2 \ddot{e} - \frac{c_2}{c_4} \dot{v} + \frac{c_2 k_3}{c_4} \dot{e} + \frac{c_2 k_1}{c_4} \dot{e} - \frac{c_2 k_1}{c_4} \frac{\sigma}{c_2} + \frac{c_2 k_1 k_3}{c_4} \frac{e}{c_2} + k_3 \dot{e}
 \end{aligned}$$

$$\Rightarrow \sigma + \left(\frac{c_2 + c_4}{c_4} \right) \left(\frac{c_4}{k_1} \right) \dot{v} = \frac{c_2 c_4}{k_1} \ddot{e} + \left(\frac{c_2 k_3}{k_1} + c_2 + c_4 \frac{k_3}{k_1} \right) \dot{e} + k_3 e$$

checks out with previous result. \leftarrow

Alternative - II

Use (a-h) but in L.T. (Laplace transform domain). Then eliminate E_i, Σ_i in favor of E & Σ .

Let $\mathcal{L}(v_i) = \Sigma_i(s), \mathcal{L}(e_i) = E_i(s)$.

$$\begin{aligned}
 \Sigma &= \Sigma_3 + \Sigma_2 = \Sigma_3 + \Sigma_1 + \Sigma_4 = k_3 E_3 + k_1 E_1 + s c_4 E_4 \\
 &= k_3 E + k_1 (E - E_2) + s c_4 (E - E_2) \\
 &= (k_3 + k_1) E + s c_4 E - (k_1 + s c_4) E_2 \\
 &= (k_3 + k_1) E + s c_4 E - (k_1 + s c_4) \frac{\Sigma_{12}}{s c_2} \\
 &= (k_3 + k_1) E + s c_4 E - (k_1 + s c_4) \frac{(\Sigma_1 - \Sigma_{13})}{s c_2} \rightarrow k_3 E_3 \rightarrow E
 \end{aligned}$$

- $E_1 + E_2 = E \checkmark$
- $E_3 = E \checkmark$
- $E_1 - E_2 = 0 \checkmark$
- $\Sigma_1 + \Sigma_4 - \Sigma_2 = 0 \checkmark$
- $\Sigma_2 + \Sigma_4 - \Sigma_1 = 0$
- $\Sigma_1 - k_1 E_1 = 0 \checkmark$
- $\Sigma_2 - s c_2 E_2 = 0 \checkmark$
- $\Sigma_3 - k_3 E_3 = 0 \checkmark$
- $\Sigma_4 - s c_4 E_4 = 0 \checkmark$

$$\Rightarrow s c_2 \Sigma = s c_2 (k_3 + k_1) E + s^2 c_2 c_4 E - (k_1 + s c_4) (\Sigma - k_3 E)$$

\mathcal{L}^{-1} gives,

$$c_2 \dot{\sigma} = c_2 (k_3 + k_1) \dot{e} + c_2 c_4 \ddot{e} - k_1 \sigma - c_4 \dot{\sigma} + k_1 k_3 e + c_4 k_3 \dot{e}$$

$$\Rightarrow \sigma + \frac{c_2 + c_4}{k_1} \dot{\sigma} = k_3 e + \left[c_2 + \frac{k_3}{k_1} (c_2 + c_4) \right] \dot{e} + \frac{c_2 c_4}{k_1} \ddot{e} \rightarrow \text{same as before}$$

Creep response: Take Laplace transform with zero initial conditions. (3)

$$\Sigma (1+As) = E (B + Cs + Ds^2)$$

$$A = \frac{c_2 + c_4}{k_1}, \quad B = k_3, \quad C = c_2 + \frac{k_3}{k_1} (c_2 + c_4), \quad D = \frac{c_2 c_4}{k_1}$$

$$\text{Now } \Sigma = \frac{\sigma_0}{s}$$

$$\Rightarrow E = \sigma_0 \frac{1}{s} \frac{(1+As)}{(B+Cs+Ds^2)}$$

Examine roots of $Ds^2 + Cs + B = 0$

$$\begin{aligned} C^2 - 4DB &= c_2^2 + \frac{k_3^2}{k_1^2} (c_2 + c_4)^2 + 2c_2 \frac{k_3}{k_1} (c_2 + c_4) - 4 \frac{k_3}{k_1} c_2 c_4 \\ &= c_2^2 + \frac{k_3^2}{k_1^2} (c_2 - c_4)^2 + 2c_2 \frac{k_3}{k_1} (c_2 - c_4) + 4 \frac{k_3^2}{k_1^2} c_2 c_4 \\ &= \left[c_2 + \frac{k_3}{k_1} (c_2 - c_4) \right]^2 + 4 \frac{k_3^2}{k_1^2} c_2 c_4 > 0 \end{aligned}$$

Thus $\sqrt{C^2 - 4DB}$ is real and $< C$.

\Rightarrow roots of $Ds^2 + Cs + B = 0$ are real and negative.

i.e., $Ds^2 + Cs + B = D(s+a)(s+b)$ where $(a, b) > 0$.

$$\text{i.e., } a, b = \frac{-C \pm \sqrt{C^2 - 4BD}}{2D}$$

\Rightarrow from L.T. tables, $e(t) = \mathcal{L}^{-1} \left\{ \frac{\sigma_0}{D} \left(\frac{1}{s(s+a)(s+b)} + \frac{A}{(s+a)(s+b)} \right) \right\}$

$$e(t) = \frac{\sigma_0}{D} \left(\frac{1}{ab} \left[1 + \frac{1}{a-b} (b e^{-at} - a e^{-bt}) \right] + \frac{A}{b-a} \left[\frac{e^{-at} - e^{-bt}}{a-b} \right] \right)$$

Creep response \rightarrow $e(t) = \frac{\sigma_0}{D} \left(\frac{1}{ab} + \frac{1}{a-b} \left(e^{-at} \left[\frac{1}{a} - A \right] + e^{-bt} \left[A - \frac{1}{b} \right] \right) \right) \blacktriangleleft$

$$e(0^+) = \frac{\sigma_0}{D} \left(\frac{1}{ab} + \frac{1}{a-b} \left(\frac{1}{a} - A + A - \frac{1}{b} \right) \right) = \frac{\sigma_0}{D} \left(\frac{1}{ab} - \frac{1}{ab} \right) = 0$$

check: (i) Initial value theorem gives,

$$e(0) = \lim_{s \rightarrow \infty} s E(s) = \lim_{s \rightarrow \infty} \sigma_0 \frac{(1+As)}{(B+Cs+Ds^2)} = 0$$

(ii) Physically, we see that \because instantaneous strains in dashpots is zero, the instantaneous overall strain is zero.

P-2 Displacements:

$$\epsilon_x = -\nu \frac{\sigma_z}{E} = \frac{\nu x}{R_x} - \frac{\nu y}{R_y} = \frac{du}{dx}$$

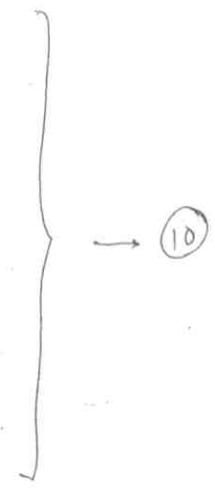
$$\epsilon_y = -\nu \frac{\sigma_z}{E} = \frac{\nu x}{R_x} - \frac{\nu y}{R_y} = \frac{dv}{dy}$$

$$\epsilon_z = \frac{\sigma_z}{E} = \frac{-x}{R_x} + \frac{y}{R_y} = \frac{dw}{dz}$$

$$\gamma_{xz} = 0 = \frac{du}{dz} + \frac{dw}{dx}$$

$$\gamma_{yx} = 0 = \frac{dv}{dy} + \frac{dw}{dx}$$

$$\gamma_{zy} = 0 = \frac{dv}{dz} + \frac{dw}{dy}$$



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(10c) → w = -xz / R_x + yz / R_y + g(x,y) → (i)

(10d, i) → ∂u/∂z = z / R_x - ∂g/∂x → u = z^2 / 2R_x - z ∂g/∂x + f(x,y) → (ii)

(10f, i) → ∂v/∂z = -z / R_y - ∂g/∂y → v = -z^2 / 2R_y - z ∂g/∂y + h(x,y) → (iii)

(10e, ii, iii) → -2z ∂^2g/∂x∂y + ∂f/∂y + ∂h/∂x = 0 → ∂^2g/∂x∂y = 0, ∂f/∂y = -∂h/∂x → (iv, v)

(10a, ii, iii) → -z ∂^2g/∂x^2 + ∂f/∂x = νx / R_x - νy / R_y → ∂^2g/∂x^2 = 0 → (vi)

∂f/∂x = νx / R_x - νy / R_y ⇒ f = νx^2 / 2R_x - νxy / R_y + m(y) → (vii)

(10b, ii, iii) → -z ∂^2g/∂y^2 + ∂h/∂y = νx / R_x - νy / R_y → ∂^2g/∂y^2 = 0 → (viii)

∂h/∂y = νx / R_x - νy / R_y ⇒ h = νxy / R_x - νy^2 / 2R_y + n(x) → (ix)

(v, vii, ix) → -νx / R_y + ∂m/∂y = -νy / R_x - ∂n/∂x

⇒ m = -νy^2 / 2R_x + C1y + C2
n = +νx^2 / 2R_y - C1x + C3

Apply bc's u=v=w=0 at (x,y,z)=(0,0,0), and rotations (∂w/∂x - ∂u/∂z), (∂w/∂y - ∂v/∂z), (∂u/∂y - ∂v/∂x) are also zero at (x,y,z)=(0,0,0) (to get rid of P.B. matrices)

Now (iv, vi, viii) → $g = c_4 x + c_5 y$

⇒ $u = \frac{z^2}{2R_x} - c_4 z + \frac{\nu}{2R_x} x^2 - \frac{\nu}{R_y} xy - \frac{\nu}{2R_x} y^2 + c_4 y + c_2$

$v = \frac{-z^2}{2R_y} - z c_5 + \frac{\nu}{R_x} xy - \frac{\nu}{2R_y} y^2 + \frac{\nu x^2}{2R_y} - c_4 x + c_5$

$w = -\frac{xz}{R_x} + \frac{yz}{R_y} + c_4 x + c_5 y$

BC'S ⇒ $c_1 = c_2 = c_3 = c_4 = c_5 = 0$ (could have got it by noting that const & linear terms in displ represent RB translation & rotation, so drop them).

⇒
$$\left. \begin{aligned} u &= \frac{1}{2R_x} [z^2 + \nu(x^2 - y^2)] - \frac{\nu}{R_y} xy \\ v &= \frac{1}{2R_y} [-z^2 + \nu(x^2 - y^2)] + \frac{\nu}{R_x} xy \\ w &= -\frac{xz}{R_x} + \frac{yz}{R_y} \end{aligned} \right\} \rightarrow (11)$$

HW #5, P. 3. Solution.

P-3 Determination of Shear Center.

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Use (25) to determine (x_{CF}, y_{CF}) as follows. So (25), (19), (22b), (23) yield,

$$\begin{aligned} x_{CF} w_y - y_{CF} w_x &= \int_A (x \tau_{yz} - y \tau_{xz}) dA \\ &= \int_A \left[x \left(-\frac{\partial \phi}{\partial x} - g(x) - \frac{1}{2} E K_y y^2 \right) - y \left(\frac{\partial \phi}{\partial y} + f(y) - \frac{1}{2} E K_x x^2 \right) \right] dA \\ &= \int_A \left[2\phi + \frac{\partial}{\partial x} [-x\phi - xy f(y) + \frac{1}{6} E K_x x^3 y] - \frac{\partial}{\partial y} [y\phi + xy g(x) + \frac{1}{6} E K_y x y^3] \right] dA \\ &= \int_A 2\phi dA + \int_C [y\phi + xy g(x) + \frac{1}{6} E K_y x y^3] dx + \int_C [-x\phi - xy f(y) + \frac{1}{6} E K_x x^3 y] dy \\ &= \int_A 2\phi dA - \frac{1}{3} E \left[K_y \int_C x y^3 dx + K_x \int_C x^3 y dy \right] \quad (\text{used (22b), (23)}) \\ &= \int_A (2\phi - E K_x x^2 y + E K_y x y^2) dA \end{aligned}$$

where ϕ is determined from (20), (22b), (23), setting $\alpha=0$. Inserting (18) in the above & equating coefficients of w_x, w_y (\because they are independent), we obtain

$$x_{CF} = \frac{1}{w_y} \int_A \left[2w_y \phi_2 + \frac{I_{xy} w_y}{F} x^2 y + \frac{I_y w_y}{F} x y^2 \right] dA \quad \left. \vphantom{x_{CF}} \right\} \rightarrow (26)$$

$$\text{where } F = I_x I_y - I_{xy}^2$$

$$\text{and } y_{CF} = -\frac{1}{w_x} \int_A \left[2w_x \phi_1 - \frac{I_x w_x}{F} x^2 y - \frac{I_{xy} w_x}{F} x y^2 \right] dA$$

In the above we have used the fact that since (20) is linear in ϕ , its solution is $\phi = w_y \phi_2 + w_x \phi_1$ where ϕ_2 and ϕ_1 are solutions of (20), (22b), (23) with $(w_y, w_x) = (1, 0)$ and $(0, 1)$, respectively. Thus ϕ_1 and ϕ_2 depend on the geometry of the cross-section only and consequently so does (x_{CF}, y_{CF}) .